

PUMP CALCULATION SHEET

SHEET 1.A.

(With considering suction and discharge equipments)

()

CLIENT : _____
 PROJECT : _____
 TITLE : _____
 JOB NO. : _____
 DOC. NO. : _____ ()

| | | | | | |
|------|---|---|---|------|--|
| REV. | 1 | 2 | 3 | MADE | |
| BY | | | | CHKD | |
| CHKD | | | | APVD | |
| APVD | | | | | |
| DATE | | | | DATE | |

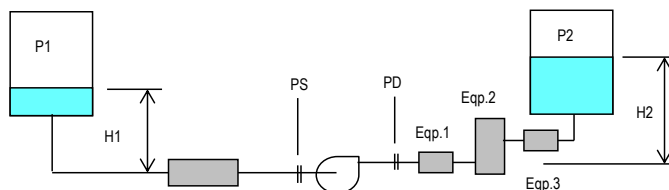
| | | | | | | |
|----|-----------------------------------|----------------------|-------------|----------------|----------|--|
| 1 | ITEM NO. : | | | | | |
| 2 | SERVICES : | | | | | |
| 3 | | FLUID | Code | Formula | | |
| 4 | PUMPED LIQUID | CONDENSATE | | | | |
| 5 | OPERATING TEMPERATURE | °C | 90 | | | Data |
| 6 | VISCOSITY AT OPERATING TEMP. | cP | 0.23 | VIS | | Data |
| 7 | VAPOR PRESSURE | kg/cm ² A | 0.15 | PV | | Data |
| 8 | SP. GR. AT OPERATING TEMP. | | 0.99 | SG | | Data |
| 9 | GRAVITY | m/s ² | 9.81 | g | | Data |
| 10 | | | | | | |
| 11 | | | | | | |
| 12 | CAPACITY | m ³ /hr | 5 | Q | | Data |
| 13 | EXCESS CAPACITY | % | 0 | X | | Data |
| 14 | MAX. CAPACITY | m ³ /hr | 5 | Qe | | Qe = 0.01(100+X)(Q) |
| 15 | | SUCTION | | | | |
| 16 | PRESSURE AT EQUIPMENT | kg/cm ² A | 1.3 | P1 | | Data |
| 17 | MIN. STATIC HEAD (+ or -) | m | -6 | H1 | | Data |
| 18 | PIPE FRICTION | kg/cm ² | 0.12 | DPp1 | | Data |
| 19 | EQUIPMENT (.....) | kg/cm ² | 0.15 | DPe1 | | Data |
| 20 | SUCTION PUMP DIA. | m | P | ID | | See Appendix, or Fill with P if by program |
| 21 | | m | 0.041 | | | Nom. Dia. by progr (inc) 1.5 ID (m)= 0.041 |
| 22 | LIQUID VELOCITY AT SUCTION NOZZLE | m/s | 1.05 | V | | V=Q/(0.25x3.14xD ² x3600) |
| 23 | VELOCITY HEAD AT SUCTION | kg/cm ² | 0.0056 | HV | | HV=0.05(SG)(V ²)/g |
| 24 | NET SUCTION PRESSURE | kg/cm ² A | 0.4304 | PS | | PS = P1+0.1(H1)(SG)-Dpe1-DPp1-HV |
| 25 | NPSH AVAILABLE | m | 2.83 | NPSHA | | NPSHA=10(Ps-PV)/SG |
| 26 | NPSH MARGIN TO NPSHR | m | 1 | | | Or may from % NPSHA, example 20 % (m) = 0.57 |
| 27 | NPSH AVAILABLE (TAKEN) | m | 1.83 | | | |
| 28 | | DISCHARGE | | | | |
| 29 | PRESSURE AT EQUIPMENT | kg/cm ² A | 2 | P2 | | Data |
| 30 | STATIC HEAD | m | 1 | H2 | | Data |
| 31 | PRESSURE DROP AT : | | | | | |
| 32 | PIPE (include fittings & valves) | kg/cm ² | 0.2 | | | Data |
| 33 | Equipment 1 (.....) | kg/cm ² | 0.11 | | | Data |
| 34 | Equipment 2 (.....) | kg/cm ² | 0.1 | | | Data |
| 35 | Equipment 3 (.....) | kg/cm ² | 0.05 | | | Data |
| 36 | Equipment 4 (.....) | kg/cm ² | 0 | | | Data |
| 37 | | | | | | |
| 38 | | | | | | |
| 39 | DISCHARGE PRESSURE DROP | kg/cm ² | 0.46 | DPp2 | | DPp2 = total discharge pressure drop |
| 40 | | | | | | |
| 41 | TOTAL DISCHARGE PRESSURE | kg/cm ² A | 2.559 | PD | | PD = P2+0.1(H2)(SG)+DpP2 |
| 42 | SUCTION PRESSURE | kg/cm ² A | 0.4304 | PS | | |
| 43 | DIFF. PRESSURE | kg/cm ² | 2.1286 | DP | | DP=PD-PS |
| 44 | TOTAL HEAD (NET) | m | 21.50 | H | | H=10(DP)/SG |
| 45 | TOTAL HEAD (TAKE) | m | 22.00 | H | | |
| 46 | LIQUID HORSE POWER | kW | 0.30 | LHP | | |
| 47 | ESTIMATED EFFICIENCY | % | 26.62 | E | | See Efficiency curve |
| 48 | ESTIMATED POWER | kW | 1.11 | BHP | | BHP=(100/E)(0.00272)(Q)(H) |
| 49 | ESTIMATED NPSHR-SINGLE SUCTION | m | 1.830 | NPSHR SS | 1.830 | 1.439 1.223 1.070 0.953 |
| 50 | ESTIMATED SPEED-SINGLE SUCTION | RPM | 2980 | | 2980 | 1480 980 730 580 |
| 51 | ESTIMATED NPSHR-DOUBLE SUCTION | m | 1.800 | NPSHR DS | 1.830 | 1.439 1.223 1.070 0.953 |
| 52 | ESTIMATED SPEED-DOUBLE SUCTION | RPM | 2980 | | 1.800 | 1.430 1.218 1.068 0.952 |
| 53 | | | | | RPM | 2980 1480 980 730 580 |
| 54 | | | | | NPSHR(m) | 1.800 1.430 1.218 1.068 0.952 |

Suction nozzle diameter

| D (inc) | ID(mm) |
|---------|--------|
| 1.5 | 0.041 |
| 2 | 0.052 |
| 3 | 0.078 |
| 4 | 0.102 |
| 6 | 0.154 |
| 8 | 0.202 |
| 10 | 0.254 |
| 12 | 0.305 |
| 14 | 0.337 |
| 16 | 0.387 |
| 18 | 0.438 |
| 20 | 0.489 |

ID based on STD thickness

REMARKS : _____



PUMP CALCULATION SHEET

SHEET 2.A

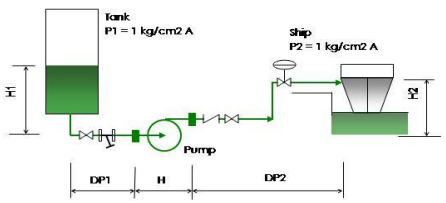
(When NPSHA is low and select vertical type)

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CLIENT : _____
 PROJECT : _____
 TITLE : _____
 JOB NO. : _____
 DOC. NO. : _____ ()

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|------|---|---|---|------|--|
| REV. | 1 | 2 | 3 | MADE | |
| BY | | | | CHKD | |
| CHKD | | | | APVD | |
| APVD | | | | | |
| DATE | | | | DATE | |

| | | | | | | |
|----|---------------------------------------|----------------------|----------|----------------------|-------------------------------|--|
| 1 | ITEM NO. : | | | | | |
| 2 | SERVICES : | | | | | |
| 3 | FLUID | | | | | |
| 4 | PUMPED LIQUID | Ammonia | | | | |
| 5 | OPERATING TEMPERATURE | °C | -33 | | | |
| 6 | VISCOSITY AT OPERATING TEMP. | cP | 0.26 | | | |
| 7 | VAPOR PRESSURE | kg/cm ² A | 1 | | | |
| 8 | SP. GR. AT OPERATING TEMP. | | 0.7 | | | |
| 9 | GRAVITY | m/s ² | 9.81 | | | |
| 10 | | | | | | |
| 11 | | | | | | |
| 12 | CAPACITY | m ³ /hr | 200 | | | |
| 13 | EXCESS CAPACITY | % | 10 | | | |
| 14 | MAX. CAPACITY | m ³ /hr | 220 | | | |
| 15 | SUCTION | | | | | |
| 16 | PRESSURE AT EQUIPMENT | kg/cm ² A | 1.025 | 1.025 | | |
| 17 | MIN. STATIC HEAD (+ or -) | m | 2 | 7 | New H1, based on impeller eye | |
| 18 | PIPE FRICTION | kg/cm ² | 0.14 | 0.14 | | |
| 19 | EQUIPMENT (.....) | kg/cm ² | 0 | 0 | | |
| 20 | SUCTION PUMP DIA. | m | p | 0.154 | Or fill with P if by program | |
| 21 | | m | 0.202 | 0.154 | D (inc) 8 8 | |
| 22 | LIQUID VELOCITY AT SUCTION NOZZLE | m/s | 1.91 | 3.28 | ID (m) 0.202 0.202 | |
| 23 | VELOCITY HEAD AT SUCTION | kg/cm ² | 0.0130 | 0.0384 | | |
| 24 | NET SUCTION PRESSURE | kg/cm ² A | 1.0120 | 1.3366 | | |
| 25 | NPSH AVAILABLE | m | 0.17 | 4.81 | | |
| 26 | DISCHARGE | | | | | |
| 27 | PRESSURE AT EQUIPMENT | kg/cm ² A | 1 | 1 | | |
| 28 | STATIC HEAD | m | 15 | 20 | | |
| 29 | PIPE (include fittings & valves) | kg/cm ² | 0.4 | 0.4 | | |
| 30 | Equipment 1 (.....CV.....) | kg/cm ² | 0.11 | 0.11 | | |
| 31 | Equipment 2 (.....) | kg/cm ² | 0 | 0 | | |
| 32 | Equipment 3 (.....) | kg/cm ² | 0 | 0 | | |
| 33 | Equipment 4 (.....) | kg/cm ² | 0 | 0 | | |
| 34 | | | | | | |
| 35 | | | | | | |
| 36 | | | | | | |
| 37 | DISCHARGE PRESSURE DROP | kg/cm ² | 0.51 | 0.51 | | |
| 38 | | | | | | |
| 39 | TOTAL DISCHARGE PRESSURE | kg/cm ² A | 2.56 | 2.91 | | |
| 40 | SUCTION PRESSURE | kg/cm ² A | 1.0120 | 1.3366 | | |
| 41 | DIFF. PRESSURE | kg/cm ² | 1.5480 | 1.5734 | | |
| 42 | TOTAL HEAD (NET) | m | 22.11 | 22.48 | | |
| 43 | TOTAL HEAD (TAKE) | m | 23 | 23 | | |
| 44 | LIQUID HORSE POWER | kW | 9.63 | 9.63 | | |
| 45 | ESTIMATED EFFICIENCY | % | 64.44 | 64.44 | See Efficiency curve | |
| 46 | ESTIMATED POWER (BHP) | kW | 14.95 | 14.95 | | |
| 47 | ESTIMATED NPSHR | m | 0.000 | 4.365 | See NPSHR curve | |
| 48 | NPSHA-NPSHR | m | 0.17 | 0.44 | | |
| 49 | ESTIMATED SPEED | RPM | | 2980 | | |
| 50 | Pump need well to give more H1 | | | Vertical centrifugal | | |
| 51 | | | | Single suction | | |
| 52 | REMARKS : | | | | | |
| 53 | Number of suction ----> | 1 | 220 | 220 | 220 | |
| 54 | 1 for single suction | | 4.365 | 2.268 | 1.607 | |
| 55 | 2 for double suction | | 2980 | 1480 | 980 | |
| 56 | | | NPSHR(m) | 4.365 | 2.268 | |
| 57 | | | | 1.607 | 1.256 | |
| 58 | | | | 1.256 | 1.027 | |
| 59 | | | | | | |
| 60 | | | | | | |



Suction nozzle diameter

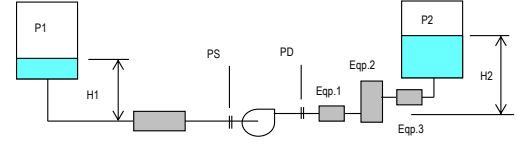
| D (inc) | ID(mm) |
|---------|--------|
| 1.5 | 0.041 |
| 2 | 0.052 |
| 3 | 0.078 |
| 4 | 0.102 |
| 6 | 0.154 |
| 8 | 0.202 |
| 10 | 0.254 |
| 12 | 0.305 |
| 14 | 0.337 |
| 16 | 0.387 |
| 18 | 0.438 |
| 20 | 0.489 |

ID based on STD thickness

PUMP CALCULATION SHEET

SHEET 5.A (More detail, with considering RPM, tip speed, head coef. & impeller dimension)
(Example: Small capacity, low head)

| | | | | |
|----|--|----------------|----------------|--|
| 1 | ITEM NO. : | | | |
| 2 | SERVICES : | | | |
| 3 | | | | |
| 4 | FLUID | Code | Formula | |
| 5 | PUMPED LIQUID | Water | | A. Plastic imp., B. With slurry, C. Dirty, D. Clean Hot, E. Clean Cold |
| 6 | LIQUID CHARACTERISTIC | D | | Data |
| 7 | OPERATING TEMPERATURE | 33 | | Data |
| 8 | VISCOSITY AT OPERATING TEMP. | 0.23 | VIS | Data |
| 9 | VAPOR PRESSURE | 0.15 | PV | Data |
| 10 | SP. GR. AT OPERATING TEMP. | 0.99 | SG | Data |
| 11 | GRAVITY | 9.81 | g | Data |
| 12 | | | | |
| 13 | CAPACITY | 5 | Q | Data |
| 14 | EXCESS CAPACITY | 30 | X | Data |
| 15 | MAX. CAPACITY | 6.5 | Qe | $Qe = 0.01(100+X)(Q)$ |
| 16 | SUCTION | | | |
| 17 | PRESSURE AT EQUIPMENT | 1.03 | P1 | Data |
| 18 | MIN. STATIC HEAD (+ or -) | -6 | H1 | Data |
| 19 | PIPE FRICTION | 0.1 | DPp1 | Data |
| 20 | EQUIPMENT(.....) | 0.05 | DPe1 | Data |
| 21 | SUCTION PUMP DIA. | 0.052 | ID | See Appendix or fill with P if by program |
| 22 | | 0.052 | | Nom. Dia. by progr (inc) 1.5 ID (m)= 0.041 |
| 23 | LIQUID VELOCITY AT SUCTION NOZZLE | 0.85 | V | $V=Q/(0.25\pi D^2 \times 3600)$ |
| 24 | VELOCITY HEAD AT SUCTION | 0.0036 | HV | $HV=0.05(SG)(V^2)/g$ |
| 25 | NET SUCTION PRESSURE | 0.2824 | PS | $PS = P1 + 0.1(H1)(SG) - DPe1 - DPp1 - HV$ |
| 26 | NPSH AVAILABLE | 1.34 | NPSHA | $NPSHA = 10(PS - PV)/SG$ |
| 27 | NPSH margin to NPSHR | 0.3 | | |
| 28 | NPSHR shall be maximum of: | 1.24 | | Max. NPSHR=NPSHA-Margin NPSH |
| 29 | DISCHARGE | | | |
| 30 | PRESSURE AT EQUIPMENT | 2 | P2 | Data |
| 31 | STATIC HEAD | 1 | H2 | Data |
| 32 | PRESSURE DROP AT : | | | |
| 33 | PIPE (include fittings & valves) | 0.1 | | Data |
| 34 | Equipment 1 (.....) | 0 | | Data |
| 35 | Equipment 2 (.....) | 0.1 | | Data |
| 36 | Equipment 3 (.....) | 0 | | Data |
| 37 | Equipment 4 (.....) | 0 | | Data |
| 38 | | | | |
| 39 | | | | |
| 40 | DISCHARGE PRESSURE DROP | 0.2 | DPp2 | DPp2 = total discharge pressure drop |
| 41 | | | | |
| 42 | TOTAL DISCHARGE PRESSURE | 2.299 | PD | $PD = P2 + 0.1(H2)(SG) + DPp2$ |
| 43 | SUCTION PRESSURE | 0.2824 | PS | |
| 44 | DIFF. PRESSURE | 2.0166 | DP | $DP = PD - PS$ |
| 45 | TOTAL HEAD (NET) | 20.37 | H | $H = 10(DP)/SG$ |
| 46 | TOTAL HEAD (TAKE) | 21.00 | H | |
| 47 | LIQUID HORSE POWER | 0.37 | | |
| 48 | 1 st Impeller Type | SINGLE SUCTION | DOUBLE SUCTION | |
| 49 | Estimated Speed | 2980 | 2980 | |
| 50 | Estimated Higher NPSHR | 0.69 | 0.437 | AT NSS = 10000 (M3/H, RPM, M) |
| 51 | Estimated Lower NPSHR | 0.40 | 0.25 | AT NSS = 15000 (M3/H, RPM, M) |
| 52 | Maximum tip speed, Umax | 55 | 55 | Note 1 |
| 53 | Approached tip speed, U | 17 | 17 | Note 1 |
| 54 | Approached head coefficient, Y | 0.765 | 0.765 | Trial with 0.4 up to 0.9 |
| 55 | Max. head per impeller, Hmax | 22.54 | 22.54 | $H_{max} = Y \cdot U^2 / g$ |
| 56 | Number of impeller, ni | 1 | 1 | |
| 57 | Average head each impeller, Hi | 21 | 21 | |
| 58 | Tip speed, Us | 16.41 | 16.41 | |
| 59 | Average tip diameter, D2 | 105.2 | 105.2 | |
| 60 | | | | |
| 61 | Cap. at each impeller eye of 1 st impelle | 6.5 | 3.25 | |
| 62 | Specific speed of first impeller | 774 | 548 | |
| 63 | Calculated 1 st impeller efficiency | 34.58 | 20.30 | |
| 64 | Specific speed of next impellers | 0 | 0 | |
| 65 | Calculated next impellers efficiency | 0.00 | 0.00 | |
| 66 | Average efficiency | 34.58 | 20.30 | |
| 67 | ESTIMATED POWER | 1.06 | 1.81 | |
| 68 | REMARKS : | | | |
| 69 | | | | |
| 70 | | | | |
| 71 | | | | |
| 72 | | | | |
| 73 | | | | |
| 74 | | | | |
| 75 | | | | |
| 76 | | | | |
| 77 | | | | |



Calculation steps:

- Calculate NPSHA
- Select NPSHR in table (by program)
- Provide RPM (by program)
- Decide U and Y (manual input)
- Decide Flow coef Cq (manual input)
- Calculate Diameter, number of stage, exit angle, tip width, efficiency & BHP (by program)

Suction nozzle diameter

| D (inc) | ID (mm) |
|---------|---------|
| 1.5 | 0.041 |
| 2 | 0.052 |
| 3 | 0.078 |
| 4 | 0.102 |
| 6 | 0.154 |
| 8 | 0.202 |
| 10 | 0.254 |
| 12 | 0.305 |
| 14 | 0.337 |
| 16 | 0.387 |
| 18 | 0.438 |
| 20 | 0.489 |

ID based on STD thickness

Speed can be modified as per motor hertz (blue fonts) or higher speed at last row (yellow)

| Speed | NPSHR range (in meter) | | | | Speed |
|-------|------------------------|-------|----------------|-------|-------|
| | Single Suction | | Double Suction | | |
| | Higher | Lower | Higher | Lower | |
| 0 | 0 | 0 | 0 | 0 | 0 |
| 580 | 0.078 | 0.048 | 0.049 | 0.029 | 580 |
| 730 | 0.106 | 0.062 | 0.067 | 0.039 | 730 |
| 980 | 0.157 | 0.092 | 0.099 | 0.058 | 980 |
| 1480 | 0.273 | 0.159 | 0.172 | 0.100 | 1480 |
| 2980 | 0.693 | 0.404 | 0.437 | 0.254 | 2980 |
| | 0.693 | 0.404 | 0.437 | 0.254 | 2980 |

Estimation result (basen on input data)

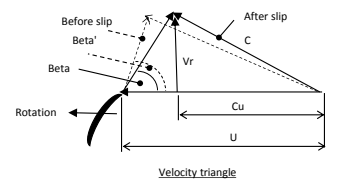
| Single Suction | | Double Suction | |
|----------------|-------------|----------------|-------------|
| Speed | 2980 RPM | at speed | 2980 RPM |
| Higher NPSHR | 0.693 meter | Higher NPSHR | 0.693 meter |
| Lower NPSHR | 0.404 meter | Lower NPSHR | 0.404 meter |
| Double Suction | | | |
| Speed | 2980 RPM | at speed | 2980 RPM |
| Higher NPSHR | 0.437 meter | Higher NPSHR | 0.437 meter |
| Lower NPSHR | 0.254 meter | Lower NPSHR | 0.254 meter |

Note 1: When at max tip speed, number of stage (line 56) =1, try to reduce approached tip speed (line 53) until max. head per impeller (line 545 almost same or just higher than average head of each impeller (line 57)

Image of impeller

| | Single suction | Double suction. Each side | |
|--|----------------|---------------------------|----------|
| Flow coef., Cq | 0.1 | 0.1 | 0.1 |
| No of blade, Z | 8 | 8 | 8 |
| Blade thickness (mm), t | 3 | 3 | 3 |
| Capacity, Qms (m3/s) | 0.00180556 | 0.00180556 | 0.000903 |
| Head coef., Y | 0.765 | 0.765 | 0.765 |
| Tangential vel. U (m/s) | 16.41 | 16.41 | 16.41 |
| Head each impella, H (m) | 21.00 | 21.00 | 21.00 |
| Cu2 (m/s) as per selected Y | 12.55 | 12.55 | 12.55 |
| Radial vel as per selected Cq, Vr (m/s) | 1.64 | 1.64 | 1.64 |
| (U-Cu) | 3.86 | 3.86 | 3.86 |
| Vr / (U-Cu) | 0.426 | 0.426 | 0.426 |
| Impeller tip dia. D (mm) | 105.2 | 105.2 | 105.2 |
| Exit angle (Beta) after slip (deg) | 23 | 23 | 23 |
| First trial Slip factor, S1 | 0.794 | 0.794 | 0.794 |
| First trial Cu' 1 (m/s) | 15.80 | 15.80 | 15.80 |
| First trial Vr / (U-Cu)'1 | 2.706 | 2.706 | 2.706 |
| First trial Exit angle (Beta1) before slip (deg) | 69.7 | 69.7 | 69.7 |
| Slip factor, S | 0.765 | 0.765 | 0.765 |
| Cu' (m/s) | 16.40 | 16.40 | 16.40 |
| Vr / (U-Cu) | 182.95 | 182.95 | 182.95 |
| Exit angle (Beta') before slip (deg) | 90 | 90 | 90 |
| Radial blade thickness, Lb (mm) | 24.0 | 24.0 | 24.0 |
| Exit area, A2 = Q/Vr (m2) | 0.00110027 | 0.00110027 | 0.00055 |
| Exit width, b (mm) | 3.6 | 3.6 | 1.8 |

Input, 0.05-0.1 for small, 0.1 - 0.2 for larger pump
Qms = Capacity in m3/s



Wiesner eq.
Backward up to radial exit
 $Lb = Z \cdot T / \sin(\beta)$

Cq will effect to exit angle and width of impeller tip exit design. $Cq = Vr / U$

PUMP CALCULATION SHEET

SHEET 5.B (More detail, with considering RPM, tip speed, head coef. & impeller dimension)
(Example: Medium capacity, high head)

| | | | | |
|----|------------------------------|----------------------|-------------|--|
| 1 | ITEM NO. : | | | |
| 2 | SERVICES : | | | |
| 3 | | | | |
| 4 | | FLUID | Code | Formula |
| 5 | PUMPED LIQUID | Hot Water | | |
| 6 | LIQUID CHARACTERISTIC | D | | A, Plastic, B, With slury, C, Dirty, D, Clean Hot, E, Clean Cold |
| 7 | OPERATING TEMPERATURE | °C | 103 | Data |
| 8 | VISCOSITY AT OPERATING TEMP. | cP | 0.27 | VIS Data |
| 9 | VAPOR PRESSURE | kg/cm ² A | 1.233 | PV Data |
| 10 | SP. GR. AT OPERATING TEMP. | | 0.955 | SG Data |
| 11 | GRAVITY | m/s ² | 9.81 | Data |
| 12 | | | | |
| 13 | | | | |
| 14 | CAPACITY | m ³ /hr | 143 | Q Data |
| 15 | EXCESS CAPACITY | % | 0 | X Data |
| 16 | MAX. CAPACITY | m ³ /hr | 143 | Qe = 0.01(100+X)(Q) |
| 17 | | | | |
| 18 | | | | |
| 19 | | | | |
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Calculation steps:

| |
|---|
| Calculate NPSHA |
| Select NPSHR in table (by program) |
| Provide RPM (by program) |
| Decide U and Y (manual input) |
| Decide Flow coef Cq (manual input) |
| Calculate Diameter, number of stage, exit angle, tip width, efficiency & BHP (by program) |

| Suction nozzle diameter | |
|-------------------------|--------|
| D (inc) | ID(mm) |
| 1.5 | 0.041 |
| 2 | 0.052 |
| 3 | 0.078 |
| 4 | 0.102 |
| 6 | 0.154 |
| 8 | 0.202 |
| 10 | 0.254 |
| 12 | 0.305 |
| 14 | 0.337 |
| 16 | 0.387 |
| 18 | 0.438 |
| 20 | 0.489 |

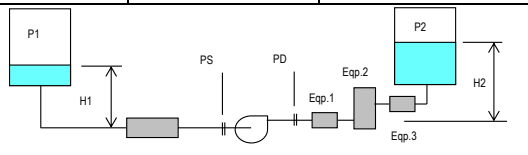
Speed can be modified as per motor hertz (blue fonts) or higher speed at last row (yellow)

| NPSHR range (in meter) | | | | | |
|---|----------------|-------|----------------|-------|-------|
| Speed | Single Suction | | Double Suction | | Speed |
| | Higher | Lower | Higher | Lower | |
| 0 | 0 | 0 | 0 | 0 | 0 |
| 580 | 0.614 | 0.358 | 0.387 | 0.225 | 580 |
| 730 | 0.834 | 0.486 | 0.526 | 0.306 | 730 |
| 980 | 1.236 | 0.720 | 0.778 | 0.453 | 980 |
| 1480 | 2.141 | 1.247 | 1.349 | 0.785 | 1480 |
| 2980 | 5.443 | 3.170 | 3.429 | 1.997 | 2980 |
| | 5.443 | 3.170 | 3.429 | 1.997 | 2980 |
| Estimation result (basen on input data) | | | | | |
| Single Suction | | | | | |
| Speed | 2980 | RPM | at speed | | 2980 |
| Higher NPSHR | 5.443 | meter | Higher NPSHR | | 5.443 |
| Lower NPSHR | 3.170 | meter | Lower NPSHR | | 3.170 |
| Double Suction | | | | | |
| Speed | 2980 | RPM | at speed | | 2980 |
| Higher NPSHR | 3.429 | meter | Higher NPSHR | | 3.429 |
| Lower NPSHR | 1.997 | meter | Lower NPSHR | | 1.997 |

Note 2: When at max tip speed, number of stage (line 56) more than 1, keep approached tip speed (line 53) same to max tip speed (line 52) or adjust lower value with following consideration :
Much number of impeller will be more expensive but give higher efficiency
Less number of impeller will be less expensive but give lower efficiency.

| Image of impeller | Single suction | | Double suction, Each side | | Input, 0.05-0.1 for small pump, 0.1 - 0.2 for larger pump |
|---|----------------|------------|---------------------------|---------|---|
| | 0.15 | 0.15 | 0.15 | 0.15 | |
| Flow coef., Cq | 0.15 | 0.15 | 0.15 | 0.15 | Input, 0.05-0.1 for small pump, 0.1 - 0.2 for larger pump |
| No of blade, Z | 6 | 6 | 6 | 6 | Input |
| Blade thickness(mm), t | 5 | 5 | 5 | 5 | Input |
| Capacity, Qms (m ³ /s) | 0.03972222 | 0.03972222 | 0.019861 | 0.01986 | Qms = Capacity in m ³ /s |
| Head coef., Y | 0.55 | 0.55 | 0.55 | 0.55 | |
| Tangensial vel, U (m/s) | 39.98 | 39.98 | 39.98 | 39.98 | |
| Head each impelle, H (m) | 89.63 | 89.63 | 89.63 | 89.63 | |
| Cu2 (m/s) as per selected Y | 21.99 | 21.99 | 21.99 | 21.99 | Cu = Y . U |
| Radial vel as per selected Cq, Vr (m/s) | 6.00 | 6.00 | 6.00 | 6.00 | Vr = Cq . U |
| (U-Cu) | 17.99 | 17.99 | 17.99 | 17.99 | |
| Vr / (U-Cu) | 0.333 | 0.333 | 0.333 | 0.333 | |
| Impeller tip dia, D (mm) | 256.2 | 256.2 | 256.2 | 256.2 | |
| Exit angle (Beta) after slip (deg) | 18 | 18 | 18 | 18 | |
| First trial Slip factor, S1 | 0.770 | 0.770 | 0.770 | 0.770 | |
| First trial Cu' 1(m/s) | 28.57 | 28.57 | 28.57 | 28.57 | |
| First trial Vr / (U-Cu)1 | 0.525 | 0.525 | 0.525 | 0.525 | |
| First trial Exit angle (Beta'1) before slip (deg) | 27.7 | 27.7 | 27.7 | 27.7 | |
| Slip factor, S | 0.728 | 0.728 | 0.728 | 0.728 | Wiesner eq. |
| Cu' (m/s) | 30.22 | 30.22 | 30.22 | 30.22 | |
| Vr / (U-Cu) | 0.61 | 0.61 | 0.61 | 0.61 | Backward up to radial exit |
| Exit angle (Beta') before slip (deg) | 32 | 32 | 32 | 32 | |
| Radial blade thickness, Lb (mm) | 57.3 | 57.3 | 57.3 | 57.3 | Lb = Z . T / sin (beta) |
| Exit area, A2 = Q/Vr (m ²) | 0.0066233 | 0.0066233 | 0.003312 | 0.00331 | |
| Exit width, b (mm) | 8.9 | 8.9 | 4.4 | 4.4 | |

Cq will effect to exit angle and width of impeller tip exit design. Cq = Vr / U



PUMP CALCULATION SHEET

SHEET 5.C [\[More detail, with considering RPM, tip speed, head coef. & impeller dimension\]](#)
[\[Example: Medium capacity high head\]](#)

| | | | | |
|----|------------------------------|--------------|-------------|---|
| 1 | ITEM NO. : | | | |
| 2 | SERVICES : | | | |
| 3 | | | | |
| 4 | | FLUID | Code | Formula |
| 5 | PUMPED LIQUID | 8FW | | |
| 6 | LIQUID CHARACTERISTIC | D | | A. Plastic, B. With slurry, C. Dirty, D. Clean Hot, E. Clean Cold |
| 7 | OPERATING TEMPERATURE | 130 | | Data |
| 8 | VISCOSITY AT OPERATING TEMP. | 0.2 | VIS | Data |
| 9 | VAPOR PRESSURE | 2.78 | PV | Data |
| 10 | SP. GR. AT OPERATING TEMP. | 0.934 | SG | Data |
| 11 | GRAVITY | 9.81 | g | Data |
| 12 | | | | |
| 13 | | | | |
| 14 | CAPACITY | 334 | Q | Data |
| 15 | EXCESS CAPACITY | 0 | X | Data |
| 16 | MAX. CAPACITY | 334 | Qe | Qe = 0.01(100+X)(Q) |
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Calculation steps:

Calculate NPSHA

Select NPSHR in table (by program)

Provide RPM (by program)

Decide U and Y (manual input)

Decide Flow coef Cq (manual input)

Calculate Diameter, number of stage, exit angle, tip width, efficiency & BHP (by program)

| Suction nozzle diameter | |
|-------------------------|---------|
| D (inc) | ID (mm) |
| 1.5 | 0.041 |
| 2 | 0.052 |
| 3 | 0.078 |
| 4 | 0.102 |
| 6 | 0.154 |
| 8 | 0.202 |
| 10 | 0.254 |
| 12 | 0.305 |
| 14 | 0.337 |
| 16 | 0.387 |
| 18 | 0.438 |
| 20 | 0.489 |

Speed can be modified as per motor hertz (blue fonts) or higher speed at last row (yellow)

| NPSHR range (in meter) | | | | | |
|------------------------|----------------|-------|----------------|-------|-------|
| Speed | Single Suction | | Double Suction | | Speed |
| | Higher | Lower | Higher | Lower | |
| 0 | 0 | 0 | 0 | 0 | 0 |
| 580 | 1.081 | 0.629 | 0.681 | 0.397 | 580 |
| 730 | 1.469 | 0.855 | 0.925 | 0.539 | 730 |
| 980 | 2.175 | 1.267 | 1.370 | 0.798 | 980 |
| 1480 | 3.768 | 2.195 | 2.374 | 1.383 | 1480 |
| 2980 | 9.581 | 5.580 | 6.036 | 3.515 | 2980 |
| | 9.581 | 5.580 | 6.036 | 3.515 | 2980 |

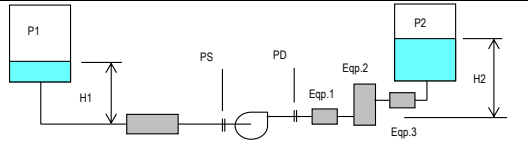
Estimation result (basen on input data)

| Single Suction | | | |
|----------------|-------|-------|--------------|
| Speed | 1480 | RPM | at speed |
| Higher NPSHR | 3.768 | meter | Higher NPSHR |
| Lower NPSHR | 2.195 | meter | Lower NPSHR |
| | | | 2980 RPM |
| | | | 9.581 meter |
| | | | 5.580 meter |
| Double Suction | | | |
| Speed | 2980 | RPM | at speed |
| Higher NPSHR | 6.036 | meter | Higher NPSHR |
| Lower NPSHR | 3.515 | meter | Lower NPSHR |
| | | | 2980 RPM |
| | | | 6.036 meter |
| | | | 3.515 meter |

Note 2: When at max tip speed, number of stage (line 56) more than 1, keep approached tip speed (line 53) same to max tip speed (line 52) or adjust lower value with following consideration :
 Much number of impeller will be more expensive but give higher efficiency
 Less number of impeller will be less expensive but give lower efficiency.

| Image of impeller | Single suction | | Double suction. Each side | | |
|---|----------------|------------|---------------------------|---------|---|
| | 0.17 | 0.17 | 0.17 | 0.17 | |
| Flow coef., Cq | 0.17 | 0.17 | 0.17 | 0.17 | Input, 0.05-0.1 for small pump, 0.1 - 0.2 for larger pump |
| No of blade, Z | 3 | 8 | 8 | 8 | Input |
| Blade thickness(mm), t | 5 | 5 | 5 | 5 | Input |
| Capacity, Qms (m3/s) | 0.09277778 | 0.09277778 | 0.046389 | 0.04639 | Qms = Capacity in m3/s |
| Head coef., Y | 0.55 | 0.55 | 0.55 | 0.55 | |
| Tangential vel, U (m/s) | 52.79 | 52.79 | 52.79 | 52.79 | |
| Head each impelle, H (m) | 156.22 | 156.22 | 156.22 | 156.22 | |
| Cu2 (m/s) as per selected Yr | 29.03 | 29.03 | 29.03 | 29.03 | Cu = Y.U |
| Radial vel as per selected Cq, Vr (m/s) | 8.97 | 8.97 | 8.97 | 8.97 | Vr = Cq . U |
| (U-Cu) | 23.75 | 23.75 | 23.75 | 23.75 | |
| Vr / (U-Cu) | 0.378 | 0.378 | 0.378 | 0.378 | |
| Impeller tip dia, D (mm) | 681.2 | 681.2 | 338.3 | 338.3 | |
| Exit angle (Beta) after slip (deg) | 21 | 21 | 21 | 21 | |
| First trial Slip factor, S1 | 0.758 | 0.758 | 0.758 | 0.758 | |
| First trial Cu' 1(m/s) | 38.30 | 38.30 | 38.30 | 38.30 | |
| First trial Vr / (U-Cu')1 | 0.619 | 0.619 | 0.619 | 0.619 | |
| First trial Exit angle (Beta'1) before slip (deg) | 31.8 | 31.8 | 31.8 | 31.8 | |
| Slip factor, S | 0.767 | 0.767 | 0.767 | 0.767 | Wiesner eq. |
| Cu' (m/s) | 37.87 | 37.87 | 37.87 | 37.87 | |
| Vr / (U-Cu') | 0.60 | 0.60 | 0.60 | 0.60 | Backward up to radial exit |
| Exit angle (Beta') before slip (deg) | 31 | 31 | 31 | 31 | |
| Radial blade thickness, Lb (mm) | 77.6 | 77.6 | 77.6 | 77.6 | Lb = Z . T / sin (beta) |
| Exit area, A2 = Q/Vr (m2) | 0.01033881 | 0.01033881 | 0.005169 | 0.00517 | |
| Exit width, b (mm) | 5.0 | 5.0 | 5.2 | 5.2 | |

Cq will effect to exit angle and width of impeller tip exit design. Cq = Vr / U



PUMP CALCULATION SHEET

SHEET 5.D [More detail, with considering RPM, tip speed, head coef. & impeller dimension]
[Example: Large capacity, medium head]

| | | | | |
|----|-------------------------------------|---------------|-------------|---|
| 1 | ITEM NO. : | | | |
| 2 | SERVICES : | | | |
| 3 | | | | |
| 4 | | FLUID | Code | Formula |
| 5 | PUMPED LIQUID | COOLING WATER | | |
| 6 | LIQUID CHARACTERISTIC | C | | A. Plastic, B. With slurry, C. Dirty, D. Clean Hot, E. Clean Cold |
| 7 | OPERATING TEMPERATURE °C | 33 | | Data |
| 8 | VISCOSITY AT OPERATING TEMP. cP | 0.12 | VIS | Data |
| 9 | VAPOR PRESSURE kg/cm ² A | 0.04 | PV | Data |
| 10 | SP. GR. AT OPERATING TEMP. | 0.97 | SG | Data |
| 11 | GRAVITY m/s ² | 9.81 | g | Data |
| 12 | | | | |
| 13 | | | | |
| 14 | CAPACITY m ³ /hr | 4211 | Q | Data |
| 15 | EXCESS CAPACITY % | 0 | X | Data |
| 16 | MAX. CAPACITY m ³ /hr | 4211 | Qe | Qe = 0.01(100+X)Q |
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Calculation steps:

Calculate NPSHA

Select NPSHR in table (by program)

Provide RPM (by program)

Decide U and Y (manual input)

Decide Flow coef Cq (manual input)

Calculate Diameter, number of stage, exit angle, tip width, efficiency & BHP (by program)

| D (inc) | ID (mm) |
|---------|---------|
| 1.5 | 0.041 |
| 2 | 0.052 |
| 3 | 0.078 |
| 4 | 0.102 |
| 6 | 0.154 |
| 8 | 0.202 |
| 10 | 0.254 |
| 12 | 0.305 |
| 14 | 0.337 |
| 16 | 0.387 |
| 18 | 0.438 |
| 20 | 0.489 |

Speed can be modified as per motor hertz (blue fonts) or higher speed at last row (yellow)

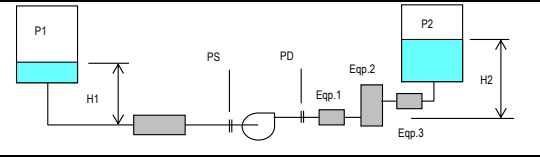
| Speed | NPSHR range (in meter) | | | | Speed |
|-------|------------------------|--------|----------------|--------|-------|
| | Single Suction | | Double Suction | | |
| | Higher | Lower | Higher | Lower | |
| 0 | 0 | 0 | 0 | 0 | 0 |
| 580 | 5.854 | 3.410 | 3.688 | 2.148 | 580 |
| 730 | 7.955 | 4.633 | 5.012 | 2.919 | 730 |
| 980 | 11.781 | 6.861 | 7.422 | 4.322 | 980 |
| 1480 | 20.413 | 11.888 | 12.860 | 7.489 | 1480 |
| 2980 | 51.900 | 30.226 | 32.695 | 19.042 | 2980 |
| | 69.884 | 40.700 | 44.025 | 25.640 | 3725 |

Estimation result (basen on input data)

| Single Suction | | Double suction, Each side | |
|----------------|-------------|---------------------------|--------------|
| Speed | 730 RPM | at speed | 3725 RPM |
| Higher NPSHR | 7.955 meter | Higher NPSHR | 69.884 meter |
| Lower NPSHR | 4.633 meter | Lower NPSHR | 40.700 meter |
| Double Suction | | | |
| Speed | 980 RPM | at speed | 3725 RPM |
| Higher NPSHR | 7.422 meter | Higher NPSHR | 44.025 meter |
| Lower NPSHR | 4.322 meter | Lower NPSHR | 25.640 meter |

| Image of impeller | Single suction | | Double suction, Each side | | |
|--|----------------|------------|---------------------------|----------|---|
| Flow coef., Cq | 0.2 | 0.2 | 0.2 | 0.2 | Input, 0.05-0.1 for small pump, 0.1 - 0.2 for larger pump |
| No of blade, Z | 6 | 6 | 6 | 6 | Input |
| Blade thickness(mm), t | 5 | 5 | 5 | 5 | Input |
| Capacity, Qms (m ³ /s) | 1.16972222 | 1.16972222 | 0.584861 | 0.584861 | Qms = Capacity in m ³ /s |
| Head coef., Y | 0.45 | 0.45 | 0.45 | 0.45 | |
| Tangensial vel. U (m/s) | 35.56 | 35.56 | 35.56 | 35.56 | |
| Head each impelle. H (m) | 58.00 | 58.00 | 58.00 | 58.00 | |
| Cu2 (m/s) as per selected Y | 16.00 | 16.00 | 16.00 | 16.00 | Cu = Y.U |
| Radial vel as per selected Cq, Vr (m/s) | 7.11 | 7.11 | 7.11 | 7.11 | Vr = Cq . U |
| (U-Cu) | 19.56 | 19.56 | 19.56 | 19.56 | |
| Vr / (U-Cu) | 0.364 | 0.364 | 0.364 | 0.364 | |
| Impeller tip dia. D (mm) | 930.3 | 930.3 | 693.0 | 693.0 | |
| Exit angle (Beta) after slip (deg) | 20 | 20 | 20 | 20 | |
| First trial Slip factor, S1 | 0.739 | 0.739 | 0.739 | 0.739 | |
| First trial Cu1 (m/s) | 21.67 | 21.67 | 21.67 | 21.67 | |
| First trial Vr1 / (U-Cu)1 | 0.512 | 0.512 | 0.512 | 0.512 | |
| First trial Exit angle (Beta1) before slip (deg) | 27.1 | 27.1 | 27.1 | 27.1 | |
| Slip factor, S | 0.684 | 0.684 | 0.684 | 0.684 | Wiesner eq. |
| Cu' (m/s) | 23.40 | 23.40 | 23.40 | 23.40 | |
| Vr / (U-Cu') | 0.58 | 0.58 | 0.58 | 0.58 | Backward up to radial exit |
| Exit angle (Beta') before slip (deg) | 30 | 30 | 30 | 30 | |
| Radial blade thickness, Lb (mm) | 59.4 | 59.4 | 59.4 | 59.4 | Lb = Z . T / sin (beta) |
| Exit area, A2 = Q/Vr (m ²) | 0.16447902 | 0.16447902 | 0.08224 | 0.08224 | |
| Exit width, b (mm) | 57.4 | 57.4 | 38.8 | 38.8 | |

Cq will effect to exit angle and width of impeller tip exit design. Cq = Vr / U



PUMP CALCULATION SHEET

SHEET 5.E (More detail, with considering RPM, tip speed, head coef. & impeller dimension)
(Example: Large capacity, medium head)

| | | | | |
|----|------------------------------|---------------|-------------|---|
| 1 | ITEM NO. : | | | |
| 2 | SERVICES : | | | |
| 3 | | | | |
| 4 | | FLUID | Code | Formula |
| 5 | PUMPED LIQUID | COOLING WATER | | |
| 6 | LIQUID CHARACTERISTIC | C | | A. Plastic, B. With slurry, C. Dirty, D. Clean Hot, E. Clean Cold |
| 7 | OPERATING TEMPERATURE | 33 | | Data |
| 8 | VISCOSITY AT OPERATING TEMP. | 0.12 | VIS | Data |
| 9 | VAPOR PRESSURE | 0.04 | PV | Data |
| 10 | SP. GR. AT OPERATING TEMP. | 0.97 | SG | Data |
| 11 | GRAVITY | 9.81 | g | Data |
| 12 | | | | |
| 13 | CAPACITY | 8145 | Q | Data |
| 14 | EXCESS CAPACITY | 0 | X | Data |
| 15 | MAX. CAPACITY | 8145 | Qe | Qe = 0.01(100+X)(Q) |
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Calculation steps:

Calculate NPSHA

Select NPSHR in table (by program)

Provide RPM (by program)

Decide U and Y (manual input)

Decide Flow coef Cq (manual input)

Calculate Diameter, number of stage, exit angle, tip width, efficiency & BHP (by program)

| Suction nozzle diameter | |
|-------------------------|--------|
| D (inc) | D (mm) |
| 1.5 | 0.041 |
| 2 | 0.052 |
| 3 | 0.078 |
| 4 | 0.102 |
| 6 | 0.154 |
| 8 | 0.202 |
| 10 | 0.254 |
| 12 | 0.305 |
| 14 | 0.337 |
| 16 | 0.387 |
| 18 | 0.438 |
| 20 | 0.489 |

Speed can be modified as per motor hertz (blue fonts) or higher speed at last row (yellow)

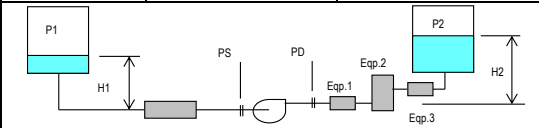
| NPSHR range (in meter) | | | | | |
|------------------------|----------------|--------|----------------|--------|-------|
| Speed | Single Suction | | Double Suction | | Speed |
| | Higher | Lower | Higher | Lower | |
| 0 | 0 | 0 | 0 | 0 | 0 |
| 580 | 9.088 | 5.293 | 5.725 | 3.334 | 580 |
| 730 | 12.350 | 7.193 | 7.780 | 4.531 | 730 |
| 980 | 18.289 | 10.652 | 11.522 | 6.710 | 980 |
| 1480 | 31.689 | 18.455 | 19.963 | 11.626 | 1480 |
| 2980 | 80.569 | 46.923 | 50.756 | 29.560 | 2980 |
| | 108.487 | 63.182 | 68.343 | 39.803 | 3725 |

Estimation result (basen on input data)

| | | | | | |
|----------------|-------|-------|--------------|---------|-------|
| Single Suction | 580 | RPM | at speed | 3725 | RPM |
| Higher NPSHR | 9.088 | meter | Higher NPSHR | 108.487 | meter |
| Lower NPSHR | 5.293 | meter | Lower NPSHR | 63.182 | meter |
| Double Suction | | | | | |
| Speed | 730 | RPM | at speed | 3725 | RPM |
| Higher NPSHR | 7.780 | meter | Higher NPSHR | 68.343 | meter |
| Lower NPSHR | 4.531 | meter | Lower NPSHR | 39.803 | meter |

| Image of impeller | Single suction | | Double suction, Each side | | Input, 0.05-0.1 for small pump, 0.1 - 0.2 for larger pump |
|---|----------------|------------|---------------------------|----------|---|
| | Flow coef., Cq | | | | |
| No of blade, Z | 10 | 10 | 10 | 10 | Input |
| Blade thickness(mm), t | 10 | 10 | 10 | 10 | Input |
| Capacity, Qms (m3/s) | 2.2625 | 2.2625 | 1.13125 | 1.13125 | Qms = Capacity in m3/s |
| Head coef., Y | 0.5 | 0.5 | 0.5 | 0.5 | |
| Tangensial vel, U (m/s) | 29.71 | 29.71 | 29.71 | 29.71 | |
| Head each impelle, H (m) | 45.00 | 45.00 | 45.00 | 45.00 | |
| Cu2 (m/s) as per selected Y | 14.86 | 14.86 | 14.86 | 14.86 | Cu = Y . U |
| Radial vel as per selected Cq, Vr (m/s) | 5.94 | 5.94 | 5.94 | 5.94 | Vr = Cq . U |
| (U-Cu) | 14.86 | 14.86 | 14.86 | 14.86 | |
| Vr / (U-Cu) | 0.400 | 0.400 | 0.400 | 0.400 | |
| Impeller tip dia, D (mm) | 978.4 | 978.4 | 777.4 | 777.4 | |
| Exit angle (Beta) after slip (deg) | 22 | 22 | 22 | 22 | |
| First trial Slip factor, S1 | 0.739 | 0.739 | 0.739 | 0.739 | |
| First trial Cu' 1(m/s) | 20.12 | 20.12 | 20.12 | 20.12 | |
| First trial Vr / (U-Cu)1 | 0.619 | 0.619 | 0.619 | 0.619 | |
| First trial Exit angle (Beta'1) before slip (deg) | 31.8 | 31.8 | 31.8 | 31.8 | |
| Slip factor, S | 0.786 | 0.786 | 0.786 | 0.786 | Wiesner eq. |
| Cu' (m/s) | 18.90 | 18.90 | 18.90 | 18.90 | |
| Vr / (U-Cu) | 0.55 | 0.55 | 0.55 | 0.55 | Backward up to radial exit |
| Exit angle (Beta') before slip (deg) | 29 | 29 | 29 | 29 | |
| Radial blade thickness, Lb (mm) | 207.6 | 207.6 | 207.6 | 207.6 | Lb = Z . T / sin (beta) |
| Exit area, A2 = Q/Vr (m2) | 0.38071749 | 0.38071749 | 0.190359 | 0.190359 | |
| Exit width, b (mm) | 132.8 | 132.8 | 85.2 | 85.2 | |

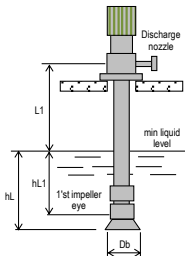
Cq will effect to exit angle and width of impeller tip exit design. Cq = Vr / U



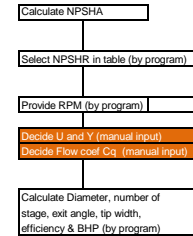
PUMP CALCULATION SHEET

SHEET 6.A [More detail, with considering RPM, tip speed, head coef. & impeller dimension]
[Vertical Pump, Submerged; Sea water pump, River water pump]

| | | | | |
|----|--|----------------|----------------|---|
| 1 | ITEM NO. : | | | |
| 2 | SERVICES : | | | |
| 3 | | | | |
| 4 | | FLUID | Code | Formula |
| 5 | PUMPED LIQUID | Sea Water | | |
| 6 | LIQUID CHARACTERISTIC | C | | A. Plastic, B. With slurry, C. Dirty, D. Clean Hot, E. Clean Cold |
| 7 | OPERATING TEMPERATURE | 32 | | Data |
| 8 | VISCOSITY AT OPERATING TEMP. | 0.23 | VIS | Data |
| 9 | VAPOR PRESSURE | 0.15 | PV | Data |
| 10 | SP. GR. AT OPERATING TEMP. | 0.99 | SG | Data |
| 11 | GRAVITY | 9.81 | g | Data |
| 12 | CAPACITY | 11000 | Q | Data |
| 13 | EXCESS CAPACITY | 0 | X | Data |
| 14 | MAX. CAPACITY | 11000 | Qe | Qe = 0.01(100+X)(Q) |
| 15 | SUCTION | | | |
| 16 | PRESSURE AT EQUIPMENT | 1.033 | P1 | Data atmospheric condition |
| 17 | LIQUID VELOCITY AT SUCTION NOZZLE | 2 | V | Data (0.61 - 2.4 m/s, recommended 1.68 m/s) |
| 18 | VELOCITY HEAD AT SUCTION | 0.0202 | HV | HV=0.05(SG)(V ² /g) |
| 19 | BELL INLET AREA | 1.5278 | Ab | Ab = (Qe/3600) / V |
| 20 | BELL INSIDE DIAMETER | 1.3947 | Db | Db = (1.2732 x Ab) ^{0.5} |
| 21 | MIN. SUBMERGENCE OF BELL | 3.129 | hL | hL = Db + 0.0002597(Qe/Db) ^{1.5} |
| 22 | MIN. SUBMERGENCE OF 1 ST IMP. EYE | 2.432 | hL1 | |
| 23 | BASE PL. OR NOZZLE TO MIN. LEVEL | 5.5 | L1 | Data |
| 24 | NET SUCTION PRESSURE | 1.2536 | PS | PS = P1+0.1(hL)(SG)-HV |
| 25 | NPSH AVAILABLE | 11.15 | NPSHA | NPSHA=10(PS-PV)/SG |
| 26 | NPSH margin to NPSHR | 2.3 | | Or from % NPSHA, typical 20% = 2.229412 m |
| 27 | NPSHR shall be maximum of: | 8.65 | | Max. NPSHR=NPSHA-Margin NPSH |
| 28 | DISCHARGE | | | |
| 29 | PRESSURE AT EQUIPMENT | 5.92 | P2 | Data |
| 30 | STATIC HEAD | 7.932 | L | L = l+hL1 |
| 31 | PRESSURE DROP AT : | | | |
| 32 | PIPE (include fittings & valves) | 0 | | Data |
| 33 | Equipment 1 (.....) | 0 | | Data |
| 34 | Equipment 2 (.....) | 0 | | Data |
| 35 | Equipment 3 (.....) | 0 | | Data |
| 36 | DISCHARGE PRESSURE DROP | 0 | DPp2 | DPp2 = total discharge pressure drop |
| 37 | | | | |
| 38 | TOTAL DISCHARGE PRESSURE | 6.71 | PD | PD = P2+0.1(L)(SG)+DPp2 (at impeller exit) |
| 39 | SUCTION PRESSURE | 1.254 | PS | |
| 40 | DIFF. PRESSURE | 5.452 | DP | DP=PD-PS |
| 41 | TOTAL HEAD (NET) | 55.07 | H | H=10(DP)/SG |
| 42 | TOTAL HEAD (TAKE) | 56 | H | |
| 43 | LIQUID HORSE POWER | 1658.76 | | |
| 44 | 1 st Impeller Type | SINGLE SUCTION | DOUBLE SUCTION | |
| 45 | Estimated Speed | 580 | 580 | |
| 46 | Estimated Higher NPSHR | - | 6.995 | AT NSS = 10000 (M3/H, RPM, M) |
| 47 | Estimated Lower NPSHR | 6.47 | 4.074 | AT NSS = 15000 (M3/H, RPM, M) |
| 48 | Maximum tip speed, U _{max} | 40 | 40 | Note 1 |
| 49 | Approached tip speed, U | 32 | 32 | Note 1 |
| 50 | Approached head coefficient, Y | 0.85 | 0.55 | Trial with 0.4 up to 0.9 |
| 51 | Max. head per impeller, H _{max} | 57.41 | 57.41 | H _{max} =Y.UA ² /g |
| 52 | Number of impeller, ni | 1 | 1 | |
| 53 | Average head each impeller, Hi | 56 | 56 | |
| 54 | Tip speed, U _i | 31.60 | 31.60 | |
| 55 | Average tip diameter, D2 | 1040.7 | 1040.7 | |
| 56 | | | | |
| 57 | Cap. at each impeller eye of 1 st impelle | 11000 | 5500 | |
| 58 | Specific speed of first impeller | 2972 | 2101 | |
| 59 | Calculated 1 st impeller efficiency | 85.58 | 84.16 | |
| 60 | Specific speed of next impellers | 0 | 0 | |
| 61 | Calculated next impellers efficiency | 0.00 | 0.00 | |
| 62 | Average efficiency | 85.58 | 84.16 | |
| 63 | ESTIMATED POWER | 1938.25 | 1971.07 | |
| 64 | REMARKS : | | | |
| 65 | | | | |
| 66 | | | | |
| 67 | | | | |
| 68 | | | | |
| 69 | | | | |
| 70 | | | | |
| 71 | | | | |
| 72 | | | | |
| 73 | | | | |
| 74 | | | | |



Calculation steps.

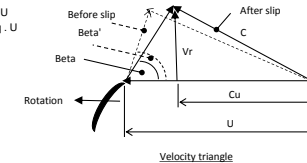


Speed can be modified as per motor hertz (blue fonts) or higher speed at last row (yellow)

| NPSHR range (in meter) | | | | | |
|---|----------------|--------|----------------|--------|--------------|
| Speed | Single Suction | | Double Suction | | Speed |
| | Higher | Lower | Higher | Lower | |
| 0 | 0 | 0 | 0 | 0 | 0 |
| 580 | 11.104 | 6.467 | 6.995 | 4.074 | 580 |
| 730 | 15.089 | 8.788 | 9.506 | 5.536 | 730 |
| 980 | 22.346 | 13.014 | 14.077 | 8.198 | 980 |
| 1480 | 38.717 | 22.549 | 24.391 | 14.205 | 1480 |
| 2980 | 98.439 | 57.330 | 62.013 | 36.116 | 2980 |
| | 98.439 | 57.330 | 62.013 | 36.116 | 2980 |
| Estimation result (basen on input data) | | | | | |
| Single Suction | | | | | |
| Speed | 580 | RPM | at speed | | 2980 RPM |
| Higher NPSHR | 0.000 | meter | Higher NPSHR | | 98.439 meter |
| Lower NPSHR | 6.467 | meter | Lower NPSHR | | 57.330 meter |
| Double Suction | | | | | |
| Speed | 580 | RPM | at speed | | 2980 RPM |
| Higher NPSHR | 6.995 | meter | Higher NPSHR | | 62.013 meter |
| Lower NPSHR | 4.074 | meter | Lower NPSHR | | 36.116 meter |

Note 1: When at max tip speed, number of stage (line 56) =1, try to reduce approached tip speed (line 53) until max. head per impeller (line 54) almost same or just higher than average head of each impeller (line 57)

| | | | |
|--|----------------|---------------------------|----------|
| Image of impeller | Single suction | Double suction. Each side | |
| Flow coef., Cq | 0.2 | 0.2 | 0.2 |
| No of blade, Z | 16 | 16 | 16 |
| Blade thickness(mm), t | 12 | 12 | 12 |
| Capacity, Qms (m3/s) | 3.0555556 | 3.0555556 | 1.527778 |
| Head coef., Y | 0.85 | 0.55 | 0.55 |
| Tangential vel. U (m/s) | 31.60 | 31.60 | 31.60 |
| Head each impelle., H (m) | 56.00 | 56.00 | 56.00 |
| Cu ² (m/s) as per selected Y | 17.38 | 17.38 | 17.38 |
| Radial vel as per selected Cq, Vr (m/s) | 6.32 | 6.32 | 6.32 |
| (U-Cu) | 14.22 | 14.22 | 14.22 |
| Vr / (U-Cu) | 0.444 | 0.444 | 0.444 |
| Impeller tip dia. D (mm) | 1040.7 | 1040.7 | 1040.7 |
| Exit angle (Beta) after slip (deg) | 24 | 24 | 24 |
| First trial Slip factor, S1 | 0.739 | 0.739 | 0.739 |
| First trial Cu ² 1/(m/s) | 23.54 | 23.54 | 23.54 |
| First trial Vr / (U-Cu)1 | 0.783 | 0.783 | 0.783 |
| First trial Exit angle (Beta1) before slip (deg) | 38.1 | 38.1 | 38.1 |
| Slip factor, S | 0.849 | 0.849 | 0.849 |
| Cu (m/s) | 20.48 | 20.48 | 20.48 |
| Vr / (U-Cu) | 0.57 | 0.57 | 0.57 |
| Exit angle (Beta) before slip (deg) | 30 | 30 | 30 |
| Radial blade thickness, Lb (mm) | 388.5 | 388.5 | 388.5 |
| Exit area, A2 = Q/Vr (m2) | 0.48340709 | 0.48340709 | 0.241704 |
| Exit width, b (mm) | 167.8 | 167.8 | 83.9 |



Wiesner eq.

Backward up to radial exit

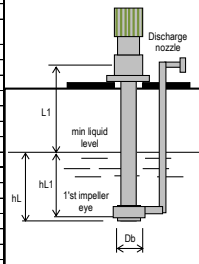
Lb = Z . T / sin (beta)

Cq will effect to exit angle and width of impeller tip exit design. Cq = Vr / U

PUMP CALCULATION SHEET

SHEET 6.B (More detail, with considering RPM, tip speed, head coef. & impeller dimension)
[Vertical Pump, Submerged: Process]

| | | | | |
|----|--|-----------------------|-----------------------------------|---|
| 1 | ITEM NO. : | | | |
| 2 | SERVICES : | | | |
| 3 | | | | |
| 4 | | FLUID | Code | Formula |
| 5 | PUMPED LIQUID | Solution | | |
| 6 | LIQUID CHARACTERISTIC | 8 | | A. Plastic, B. With slurry, C. Dirty, D. Clean Hot, E. Clean Cold |
| 7 | OPERATING TEMPERATURE | 90 | | Data |
| 8 | VISCOSITY AT OPERATING TEMP. | 0.23 | VIS | Data |
| 9 | VAPOR PRESSURE | 0.2 | PV | Data |
| 10 | SP. GR. AT OPERATING TEMP. | 0.99 | SG | Data |
| 11 | GRAVITY | 9.81 | g | Data |
| 12 | CAPACITY | 35 | Q | Data |
| 13 | EXCESS CAPACITY | 0 | X | Data |
| 14 | MAX. CAPACITY | 35 | Qe | Qe = 0.01(100+X)(Q) |
| 15 | SUCTION | | | |
| 16 | PRESSURE AT EQUIPMENT | 1.033 | P1 | Data atmospheric condition |
| 17 | LIQUID VELOCITY AT SUCTION NOZZLE | 1.5 | V | Data (0.61 - 2.4 m/s, recommended 1.68 m/s) |
| 18 | VELOCITY HEAD AT SUCTION | 0.0114 | HV | HV=0.05(SG)(V ² /g) |
| 19 | BELL INLET AREA | 0.0065 | Ab | Ab = (Qe/3600) / V |
| 20 | BELL INSIDE DIAMETER | 0.0908 | Db | Db = (1.2732 x Ab) ^{0.5} |
| 21 | MIN. SUBMERGENCE OF BELL | 0.423 | hL | hL = Db + 0.0002597(Qe/Db ^{1.5}) |
| 22 | MIN. SUBMERGENCE OF 1 ST IMP. EYE | 0.377 | hL1 | |
| 23 | BASE PL. OR NOZZLE TO MIN. LEVEL | 1.5 | L1 | Data |
| 24 | NET SUCTION PRESSURE | 1.0590 | PS | PS = P1+0.1(hL1)(SG)+HV |
| 25 | NPSH AVAILABLE | 8.68 | NPSHA | NPSHA=10(PS-PV)/SG |
| 26 | NPSH margin to NPSHR | 2.5 | | Or may from % NPSHA, example 20 % (m) = 1.74 |
| 27 | NPSHR shall be maximum of: | 6.18 | | Max. NPSHR=NPSHA-Margin NPSH |
| 28 | DISCHARGE | | | |
| 29 | PRESSURE AT EQUIPMENT | 5 | P2 | Data |
| 30 | STATIC HEAD | 1.877 | L | L = L1+hL1 |
| 31 | PRESSURE DROP AT : | | | |
| 32 | PIPE (include fittings & valves) | 0 | | Data |
| 33 | Equipment 1 (.....) | 0 | | Data |
| 34 | Equipment 2 (.....) | 0 | | Data |
| 35 | Equipment 3 (.....) | 0 | | Data |
| 36 | DISCHARGE PRESSURE DROP | 0 | DPp2 | DPp2 = total discharge pressure drop |
| 37 | TOTAL DISCHARGE PRESSURE | 5.19 | PD | PD = P2+0.1(L)(SG)+DPp2 (at impeller exit) |
| 38 | SUCTION PRESSURE | 1.059 | PS | |
| 39 | DIFF. PRESSURE | 4.127 | DP | DP=PD-PS |
| 40 | TOTAL HEAD (NET) | 41.69 | H | H=10(DP)/SG |
| 41 | TOTAL HEAD (TAKE) | 42 | H | |
| 42 | LIQUID HORSE POWER | 3.96 | | |
| 43 | 1st Impeller Type | SINGLE SUCTION | SINGLE SUCTION-LOWER SPEED | |
| 44 | Estimated Speed | 2980 | 1480 | |
| 45 | Estimated Higher NPSHR | 2.13 | 0.838 | AT NSS = 10000 (M3/H, RPM, M) |
| 46 | Estimated Lower NPSHR | 1.24 | 0.488 | AT NSS = 15000 (M3/H, RPM, M) |
| 47 | Maximum tip speed, Umax | 30 | 30 | Note 1 |
| 48 | Approached tip speed, U | 29 | 29 | Note 1 |
| 49 | Approached head coefficient, Y | 0.55 | 0.55 | Trial with 0.4 up to 0.9 |
| 50 | Max. head per impeller, Hmax | 47.15 | 47.15 | Hmax=Y.U ² /g |
| 51 | Number of impeller, ni | 1 | 1 | |
| 52 | Average head eah impeller, Hi | 42 | 42 | |
| 53 | Tip speed, Us | 27.37 | 27.37 | |
| 54 | Average tip diameter, D2 | 175.4 | 353.2 | |
| 55 | | | | |
| 56 | | | | |
| 57 | | | | |
| 58 | Cap. at each impeller eye of 1 st impelle | 35 | 35 | |
| 59 | Specific speed of first impeller | 1069 | 531 | |
| 60 | Calculated 1 st impeller efficiency | 59.49 | 50.50 | |
| 61 | Specific speed of next impellers | 0 | 0 | |
| 62 | Calculated next impellers efficiency | 0.00 | 0.00 | |
| 63 | Average efficiency | 59.49 | 50.50 | |
| 64 | ESTIMATED POWER | 6.65 | 7.84 | |
| 65 | REMARKS : | | | |
| 66 | | | | |
| 67 | | | | |
| 68 | | | | |
| 69 | | | | |
| 70 | | | | |
| 71 | | | | |
| 72 | | | | |
| 73 | | | | |
| 74 | | | | |



Calculation steps.

```

    graph TD
    A[Calculate NPSHA] --> B[Select NPSHR in table (by program)]
    B --> C[Provide RPM (by program)]
    C --> D[Decide U and Y (manual input)]
    D --> E[Decide Flow coef Cq (manual input)]
    E --> F[Calculate Diameter, number of stage, exit angle, tip width, efficiency & BHP (by program)]
  
```

Speed can be modified as per motor hertz (blue fonts) or higher speed at last row (yellow)

| NPSHR range (in meter) | | | | | |
|---|----------------|-------|----------------|-------|----------|
| Speed | Single Suction | | Double Suction | | Speed |
| | Higher | Lower | Higher | Lower | |
| 0 | 0 | 0 | 0 | 0 | 0 |
| 580 | 0.240 | 0.140 | 0.151 | 0.088 | 580 |
| 730 | 0.326 | 0.190 | 0.206 | 0.120 | 730 |
| 980 | 0.483 | 0.282 | 0.305 | 0.177 | 980 |
| 1480 | 0.838 | 0.488 | 0.528 | 0.307 | 1480 |
| 2980 | 2.130 | 1.240 | 1.342 | 0.781 | 2980 |
| | 2.130 | 1.240 | 1.342 | 0.781 | 2980 |
| Estimation result (basen on input data) | | | | | |
| Single Suction | | | | | |
| Speed | 2980 | RPM | at speed | | 2980 RPM |
| Higher NPSHR | 2.130 | meter | Higher NPSHR | 2.130 | meter |
| Lower NPSHR | 1.240 | meter | Lower NPSHR | 1.240 | meter |
| Double Suction | | | | | |
| Speed | 2980 | RPM | at speed | | 2980 RPM |
| Higher NPSHR | 1.342 | meter | Higher NPSHR | 1.342 | meter |
| Lower NPSHR | 0.781 | meter | Lower NPSHR | 0.781 | meter |

Note 1: When at max tip speed, number of stage (line 56) =1, try to reduce approached tip speed (line 53) until max. head per impeller (line 54) almost same or just higher than average head of each impeller (line 57)

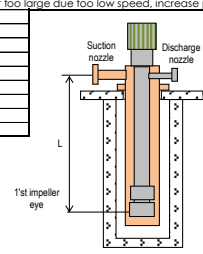
| Image of impeller | Single suction | | |
|--|----------------|------------|---|
| | 0.15 | 0.15 | |
| Flow coef., Cq | 0.15 | 0.15 | Input, 0.05-0.1 for small pump, 0.1 - 0.2 for larger pump |
| No of blade, Z | 6 | 6 | Input |
| Blade thickness(mm), t | 5 | 5 | Input |
| Capacity, Qms (m3/s) | 0.00972222 | 0.00972222 | Qms = Capacity in m3/s |
| Head coef., Y | 0.55 | 0.55 | |
| Tangensial vel. U (m/s) | 27.37 | 27.37 | |
| Head each impelle, H (m) | 42.00 | 42.00 | |
| Cu2 (m/s) as per selected Y | 15.05 | 15.05 | Cu = Y.U |
| Radial vel as per selected Cq, Vr (m/s) | 4.11 | 4.11 | Vr = Cq . U |
| (U-Cu) | 12.32 | 12.32 | |
| Vr / (U-Cu) | 0.333 | 0.333 | |
| Impeller tip dia, D (mm) | 175.4 | 175.4 | |
| Exit angle (Beta) after slip (deg) | 18 | 18 | |
| First trial Slip factor, S1 | 0.770 | 0.770 | |
| First trial Cu / (U-Cu)1 | 19.56 | 19.56 | |
| First trial Vr / (U-Cu)1 | 0.525 | 0.525 | |
| First trial Exit angle (Beta1) before slip (deg) | 27.7 | 27.7 | |
| Slip factor, S | 0.728 | 0.728 | Wiesner eq. |
| Cu' (m/s) | 20.69 | 20.69 | |
| Vr / (U-Cu') | 0.61 | 0.61 | Backward up to radial exit |
| Exit angle (Beta') before slip (deg) | 32 | 32 | |
| Radial blade thickness, Lb (mm) | 57.3 | 57.3 | Lb = Z . T / sin (beta) |
| Exit area, A2 = Q/Vr (m2) | 0.00236808 | 0.00236808 | |
| Exit width, b (mm) | 4.8 | 4.8 | |

Cq will effect to exit angle and width of impeller tip exit design. Cq = Vr / U

PUMP CALCULATION SHEET

SHEET 6.C (More detail, with considering RPM, tip speed, head coef. & impeller dimension)
 [Vertical canned pump; Process with high vapor pressure]

| | | | | |
|----|--|-----------------------|----------------|--|
| 1 | ITEM NO. : | | | |
| 2 | SERVICES : | | | |
| 3 | | FLUID | Code | Formula |
| 4 | PUMPED LIQUID | Ammonia Liq. | | |
| 5 | LIQUID CHARACTERISTIC | D | | A, Plastic, B. With slury, C. Dirty, D. Clean Hot. E. Clean Cold |
| 6 | OPERATING TEMPERATURE | °C | -33.1 | Data |
| 7 | VISCOSITY AT OPERATING TEMP. | cP | 0.272 | DATA |
| 8 | VAPOR PRESSURE | kg/cm ² A | 1.046 | DATA |
| 9 | SP. GR. AT OPERATING TEMP. | | 0.682 | DATA |
| 10 | GRAVITY | nt/s ² | 9.81 | DATA |
| 11 | | | | |
| 12 | | | | |
| 13 | CAPACITY | m ³ /hr | 1620 | Q |
| 14 | EXCESS CAPACITY | % | 0 | X |
| 15 | MAX. CAPACITY | m ³ /hr | 1620 | Qe = 0.01(100+X)(Q) |
| 16 | SUCTION | | | |
| 17 | PRESSURE AT EQUIPMENT | kg/cm ² A | 1.033 | P1 |
| 18 | MINI. STATIC HEAD (+ or -) | m | 3 | H1 |
| 19 | PIPE FRICTION LOSS | kg/cm ² | 0.02 | DPp1 |
| 20 | EQUIPMENT LOSS | kg/cm ² | 0 | DPe1 |
| 21 | PRESSURE AT SUCTION NOZZLE | kg/cm ² A | 1.2574 | |
| 22 | SUCTION PUMP DIA. | m | 0.489 | ID |
| 23 | | | 0.489 | ID |
| 24 | LIQUID VELOCITY AT SUCTION NOZZLE | m/s | 2.40 | V = Q/(0.25x3.14xD ² x3600) |
| 25 | VELOCITY HEAD AT SUCTION | kg/cm ² | 0.0200 | HV = 0.05(SG)V ² /g |
| 26 | NET SUCTION STATIC PRESSURE | kg/cm ² A | 1.2376 | PS = P1 + 0.1(H1)(SG) - DPe1 - DPp1 - HV |
| 27 | DISCHARGE | | | |
| 28 | PRESSURE AT EQUIPMENT | kg/cm ² A | 9.28 | P2 |
| 29 | STATIC HEAD | m | 12 | H2 |
| 30 | PRESSURE DROP AT : | | | |
| 31 | PIPE (include fittings & valves) | kg/cm ² | 0.3 | |
| 32 | Equipment 1 (.....) | kg/cm ² | 0.3 | |
| 33 | Equipment 2 (.....) | kg/cm ² | 0 | |
| 34 | Equipment 3 (.....) | kg/cm ² | 0 | |
| 35 | Equipment 4 (.....) | kg/cm ² | 0 | |
| 36 | DISCHARGE PRESSURE DROP | kg/cm ² | 0.2 | DPp2 = total discharge pressure drop |
| 37 | TOTAL DISCHARGE PRESSURE | kg/cm ² A | 10.30 | PD = P2 + 0.1(H2)(SG) + DPp2 AT DISCHARGE NOZZLE |
| 38 | SUCTION PRESSURE | kg/cm ² A | 1.2576 | PS |
| 39 | DIFF. PRESSURE | kg/cm ² | 9.0408 | DP = PD - PS |
| 40 | TOTAL HEAD (NET) | m | 132.56 | H = 10(DP)/SG |
| 41 | TOTAL HEAD (TAKE) | m | 133.00 | H |
| 42 | LIQUID HORSE POWER | kw | 399.69 | LHP |
| 43 | PRESSURE DROP INSIDE PUMP | kg/cm ² | 0.16 | Has to be calculated, but may be approach 8 x HV |
| 44 | TOTAL HEAD REQUIRED AT IMPELLER | m | 135.34 | Hi |
| 45 | LIQUID HORSE POWER | kw | 406.72 | LHPi |
| 46 | PUMP LENGTH | | | |
| 47 | ESTIMATED/TOTAL PUMP LENGTH | m | 9 | L |
| 48 | SUCTION IMPELLER PRESSURE | | 1.651 | |
| 49 | NPSHA AT IMPELLER EYE | m | 11.81 | |
| 50 | NPSH MARGIN | m | 0.5 | Or may from % NPSHA, example 20% (m) = 2.362007 |
| 51 | NPSHR shall be maximum of: | m | 11.31 | Max. NPSHR = NPSHA - Margin NPSH |
| 52 | 1 st Impeller type | | SINGLE SUCTION | |
| 53 | Estimated Speed | RPM | 1480 | If this speed too low try to increase pump length |
| 54 | Estimated Higher NPSHR | m | 10.80 | AT NSS = 10000 (M3/H, RPM, M) |
| 55 | Estimated Lower NPSHR | m | 6.29 | AT NSS = 15000 (M3/H, RPM, M) |
| 56 | Maximum tip speed, Umax | m/s | 55 | |
| 57 | Approached tip speed, U | m/s | 3.6 | |
| 58 | | | | |
| 59 | Approached head coefficient, Y | | 0.55 | Trial with 0.4 up to 0.9 |
| 60 | Max. head per impeller, Hmax | m | 72.66 | Hmax = Y.U ² /g |
| 61 | Number of impeller, ni | | 2 | |
| 62 | Average head eah impeller, Hi | m | 67.67 | |
| 63 | Tip speed, Us | m/s | 34.74 | |
| 64 | Average tip diameter, D2 | mm | 448.3 | If this diameter too large due too low speed, increase pump length |
| 65 | | | | |
| 66 | Cap. at each impeller eye of 1 st impelle | m ³ /h | 1620 | |
| 67 | Specific speed of first impeller | m ³ /h.rpm | 2525 | |
| 68 | Calculated 1 st impeller efficiency | % | 84.36 | |
| 69 | Specific speed of next impellers | m ³ /h.rpm | 2525 | |
| 70 | Calculated next impellers efficiency | % | 84.36 | |
| 71 | Average efficiency | % | 84.36 | At impellers |
| 72 | ESTIMATED POWER | kw | 482.13 | |
| 73 | | | | |
| 74 | REMARKS : | | | |
| 75 | | | | |
| 76 | | | | |
| 77 | | | | |
| 78 | | | | |
| 79 | | | | |
| 80 | | | | |
| 81 | | | | |
| 82 | | | | |



Calculation steps:

```

    graph TD
    A[Calculate NPSHA] --> B[Select NPSHR in table (by program)]
    B --> C[Provide RPM (by program)]
    C --> D[Decide U and Y (manual input)]
    D --> E[Decide Flow coef Cq (manual input)]
    E --> F[Calculate Diameter, number of stage, exit angle, tip width, efficiency & BHP (by program)]
    
```

Suction nozzle diameter

| D (inc) | ID(mm) |
|---------|--------|
| 1.5 | 0.041 |
| 2 | 0.052 |
| 3 | 0.078 |
| 4 | 0.102 |
| 6 | 0.154 |
| 8 | 0.202 |
| 10 | 0.254 |
| 12 | 0.305 |
| 14 | 0.337 |
| 16 | 0.387 |
| 18 | 0.438 |
| 20 | 0.489 |

ID based on STD thickness

Speed can be modified as per motor hertz (blue fonts) or higher speed at last row (yellow)

| NPSHR range (in meter) | | | | | |
|------------------------|----------------|--------|----------------|--------|-------|
| Speed | Single Suction | | Double Suction | | Speed |
| | Higher | Lower | Higher | Lower | |
| 0 | 0 | 0 | 0 | 0 | 0 |
| 580 | 3.097 | 1.804 | 1.951 | 1.136 | 580 |
| 730 | 4.208 | 2.451 | 2.651 | 1.544 | 730 |
| 980 | 6.232 | 3.629 | 3.926 | 2.286 | 980 |
| 1480 | 10.798 | 6.289 | 6.802 | 3.962 | 1480 |
| 2980 | 27.453 | 15.989 | 17.295 | 10.072 | 2980 |
| | 27.453 | 15.989 | 17.295 | 10.072 | 2980 |

Estimation result (basen on input data)

| Single Suction | | | | | |
|----------------|--------|-------|--------------|--------|-------|
| Speed | 1480 | RPM | at speed | 2980 | RPM |
| Higher NPSHR | 10.798 | meter | Higher NPSHR | 27.453 | meter |
| Lower NPSHR | 6.289 | meter | Lower NPSHR | 15.989 | meter |

| Double Suction | | | | | |
|----------------|-------|-------|--------------|--------|-------|
| Speed | 1480 | RPM | at speed | 2980 | RPM |
| Higher NPSHR | 6.802 | meter | Higher NPSHR | 17.295 | meter |
| Lower NPSHR | 3.962 | meter | Lower NPSHR | 10.072 | meter |

Image of Impeller

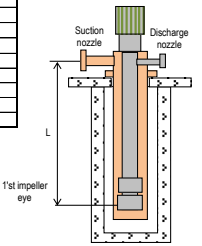
| Single suction | | |
|--|------------|------------|
| Flow coef., Cq | 0.15 | 0.15 |
| No of blade, Z | 6 | 6 |
| Blade thickness(mm), t | 5 | 5 |
| Capacity, Qrms (m3/s) | 0.45 | 0.45 |
| Head coef., Y | 0.55 | 0.55 |
| Tangential vel, U (m/s) | 34.74 | 34.74 |
| Head each impeller, H (m) | 67.67 | 67.67 |
| Cu ² (m/s) as per selected Y | 19.11 | 19.11 |
| Radial vel as per selected Cq, Vr (m/s) | 5.21 | 5.21 |
| (U-Cu) | 15.63 | 15.63 |
| Vr / (U-Cu) | 0.333 | 0.333 |
| Impeller tip dia, D (mm) | 448.3 | 448.3 |
| Exit angle (Beta) after slip (deg) | 18 | 18 |
| First trial Slip factor, S1 | 0.770 | 0.770 |
| First trial Cu ² (m/s) | 24.82 | 24.82 |
| First trial Vr / (U-Cu) | 0.525 | 0.525 |
| First trial Exit angle (Beta1) before slip (deg) | 27.7 | 27.7 |
| Slip factor, S | 0.728 | 0.728 |
| Cu' (m/s) | 26.26 | 26.26 |
| Vr / (U-Cu') | 0.61 | 0.61 |
| Exit angle (Beta') before slip (deg) | 32 | 32 |
| Radial blade thickness, Lb (mm) | 57.3 | 57.3 |
| Exit area, A2 = Q/Vr (m2) | 0.08635126 | 0.08635126 |
| Exit width, b (mm) | 63.9 | 63.9 |

Input, 0.05-0.1 for small pump, 0.1 - 0.2 for larger pump
 Orms = Capacity in m3/s
 Cu = Y.U
 Vr = Cq . U
 Wiesner eq.
 Backward up to radial exit
 Lb = Z . T / sin (beta)
 Cq will effect to exit angle and width of impeller tip exit design. Cq = Vr / U

PUMP CALCULATION SHEET

SHEET 6.D (More detail, with considering RPM, tip speed, head coef. & impeller dimension)
(Vertical canned pump: Process with high vapor pressure)

| | | | | |
|----|--|-----------------------|----------------|--|
| 1 | ITEM NO. : | | | |
| 2 | SERVICES : | | | |
| 3 | | | | |
| 4 | | FLUID | Code | Formula |
| 5 | PUMPED LIQUID | Ammonia Liq. | | |
| 6 | LIQUID CHARACTERISTIC | D | | A, Plastic, B. With slury, C. Dirty, D. Clean Hot, E. Clean Cold |
| 7 | OPERATING TEMPERATURE | °C | -33.1 | Data |
| 8 | VISCOSITY AT OPERATING TEMP. | cP | 0.272 | Data |
| 9 | VAPOR PRESSURE | kg/cm ² A | 1.046 | PV |
| 10 | SP. GR. AT OPERATING TEMP. | | 0.682 | SG |
| 11 | GRAVITY | m/s ² | 9.81 | g |
| 12 | | | | |
| 13 | CAPACITY | m ³ /hr | 1620 | Q |
| 14 | EXCESS CAPACITY | % | 0 | X |
| 15 | MAX. CAPACITY | m ³ /hr | 1620 | Qe = 0.01(100+X)(Q) |
| 16 | SUCTION | | | |
| 17 | PRESSURE AT EQUIPMENT | kg/cm ² A | 1.033 | P1 |
| 18 | MIN. STATIC HEAD (+ or -) | m | 3 | H1 |
| 19 | PIPE FRICTION LOSS | kg/cm ² | 0.02 | DPp1 |
| 20 | EQUIPMENT LOSS | kg/cm ² | 0 | DPe1 |
| 21 | PRESSURE AT SUCTION NOZZLE | kg/cm ² A | 1.2576 | |
| 22 | SUCTION PUMP DIA. | m | P | See Appendix or fill with P if by program |
| 23 | | m | 0.387 | ID |
| 24 | LIQUID VELOCITY AT SUCTION NOZZLE | m/s | 3.83 | V = Q / (0.25 x π x ID ² x 3600) |
| 25 | VELOCITY HEAD AT SUCTION | kg/cm ² | 0.0509 | HV = 0.05(SG)(V ²) / g |
| 26 | NET SUCTION STATIC PRESSURE | kg/cm ² A | 1.2067 | PS = P1 + 0.1(H1)(SG) + Dpe1 + DPp1 + HV |
| 27 | DISCHARGE | | | |
| 28 | PRESSURE AT EQUIPMENT | kg/cm ² A | 9.28 | P2 |
| 29 | STATIC HEAD | m | 12 | H2 |
| 30 | PRESSURE DROP AT : | | | |
| 31 | PIPE (include fittings & valves) | kg/cm ² | 0.3 | Data |
| 32 | Equipment 1 (.....) | kg/cm ² | 0.3 | Data |
| 33 | Equipment 2 (.....) | kg/cm ² | 0 | Data |
| 34 | Equipment 3 (.....) | kg/cm ² | 0 | Data |
| 35 | Equipment 4 (.....) | kg/cm ² | 0 | Data |
| 36 | DISCHARGE PRESSURE DROP | kg/cm ² | 0.2 | DPp2 |
| 37 | TOTAL DISCHARGE PRESSURE | kg/cm ² A | 10.30 | PD = P2 + 0.1(H2)(SG) + DPp2 AT DISCHARGE NOZZLE |
| 38 | SUCTION PRESSURE | kg/cm ² A | 1.2576 | PS |
| 39 | DIFF. PRESSURE | kg/cm ² | 9.0408 | DP = PD - PS |
| 40 | TOTAL HEAD (NET) | m | 132.56 | H = 10(DP)/SG |
| 41 | TOTAL HEAD (TAKE) | m | 133.00 | H |
| 42 | LIQUID HORSE POWER | kw | 399.69 | LHP |
| 43 | PRESSURE DROP INSIDE PUMP | kg/cm ² | 0.41 | Has to be calculated, but may be approach 8 x HV |
| 44 | TOTAL HEAD REQUIRED AT IMPELLER | m | 138.97 | |
| 45 | LIQUID HORSE POWER | kw | 417.62 | AT IMPELLERS |
| 46 | PUMP LENGTH | | | |
| 47 | ESTIMATED/TRIAL PUMP LENGTH | m | 6.7 | L |
| 48 | SUCTION IMPELLER PRESSURE | | 1.664 | From suction nozzle to first impeller eye |
| 49 | NPSHA AT IMPELLER EYE | m | 9.06 | |
| 50 | NPSH MARGIN | m | 0.5 | Or may from % NPSHA, example 20 % (m) = 1.811351 |
| 51 | NPSHR shall be maximum of: | m | 8.56 | Max. NPSHR = NPSHA - Margin NPSH |
| 52 | 1 st Impeller type | | SINGLE SUCTION | |
| 53 | Estimated Speed | RPM | 980 | If this speed too low try to increase pump length |
| 54 | Estimated Higher NPSHR | m | 4.23 | AT NSS = 10000 (M3/H, RPM, M) |
| 55 | Estimated Lower NPSHR | m | 3.43 | AT NSS = 15000 (M3/H, RPM, M) |
| 56 | Maximum tip speed, Umax | m/s | 55 | |
| 57 | Approached tip speed, U | m/s | 25 | |
| 58 | | | | |
| 59 | Approached head coefficient, Y | | 0.55 | Trial with 0.4 up to 0.9 |
| 60 | Max. head per impeller, Hmax | m | 35.04 | Hmax = Y * U ² / g |
| 61 | Number of impeller, ni | | 4 | |
| 62 | Average head each impeller, Hi | m | 34.74 | |
| 63 | Tip speed, Us | m/s | 24.89 | |
| 64 | Average tip diameter, D2 | mm | 485.1 | If this diameter too large due too low speed, increase pump length |
| 65 | | | | |
| 66 | Cap. at each impeller eye of 1 st impelle | m ³ /h | 1620 | |
| 67 | Specific speed of first impeller | m ³ /h.rpm | 2756 | |
| 68 | Calculated 1 st impeller efficiency | % | 84.63 | |
| 69 | Specific speed of next impellers | m ³ /h.rpm | 2756 | |
| 70 | Calculated next impellers efficiency | % | 84.63 | |
| 71 | Average efficiency | % | 84.63 | At Impellers |
| 72 | ESTIMATED POWER | kw | 493.46 | |
| 73 | | | | |
| 74 | REMARKS : | | | |
| 75 | | | | |
| 76 | | | | |
| 77 | | | | |
| 78 | | | | |
| 79 | | | | |
| 80 | | | | |
| 81 | | | | |
| 82 | | | | |



Calculation steps:

```

    graph TD
    A[Calculate NPSHA] --> B[Select NPSHR in table (by program)]
    B --> C[Provide RPM (by program)]
    C --> D[Decide U and Y (manual input)]
    D --> E[Decide Flow coef Cq (manual input)]
    E --> F[Calculate Diameter, number of stage, exit angle, tip width, efficiency & BHP (by program)]
  
```

| Suction nozzle diameter | D (inc) | ID (mm) |
|-------------------------|---------|---------|
| | 1.5 | 0.041 |
| | 2 | 0.052 |
| | 3 | 0.078 |
| | 4 | 0.102 |
| | 6 | 0.154 |
| | 8 | 0.202 |
| | 10 | 0.254 |
| | 12 | 0.305 |
| | 14 | 0.337 |
| | 16 | 0.387 |
| | 18 | 0.438 |
| | 20 | 0.489 |

Speed can be modified as per motor hertz (blue fonts) or higher speed at last row (yellow)

| Speed | NPSHR range (in meter) | | | | Speed |
|-------|------------------------|----------------------|-----------------------|----------------------|-------|
| | Single Suction Higher | Single Suction Lower | Double Suction Higher | Double Suction Lower | |
| 0 | 0 | 0 | 0 | 0 | 0 |
| 580 | 3.097 | 1.804 | 1.951 | 1.136 | 580 |
| 730 | 4.208 | 2.451 | 2.651 | 1.544 | 730 |
| 980 | 6.232 | 3.629 | 3.926 | 2.286 | 980 |
| 1480 | 10.798 | 6.289 | 6.802 | 3.962 | 1480 |
| 2980 | 27.453 | 15.989 | 17.295 | 10.072 | 2980 |
| | 27.453 | 15.989 | 17.295 | 10.072 | 2980 |

| Single Suction | Estimation result (basen on input data) | | | | |
|----------------|---|-------|--------------|--------|-------|
| Speed | 980 | RPM | at speed | 2980 | RPM |
| Higher NPSHR | 6.232 | meter | Higher NPSHR | 27.453 | meter |
| Lower NPSHR | 3.629 | meter | Lower NPSHR | 15.989 | meter |

| Double Suction | Estimation result (basen on input data) | | | | |
|----------------|---|-------|--------------|--------|-------|
| Speed | 1480 | RPM | at speed | 2980 | RPM |
| Higher NPSHR | 6.802 | meter | Higher NPSHR | 17.295 | meter |
| Lower NPSHR | 3.962 | meter | Lower NPSHR | 10.072 | meter |

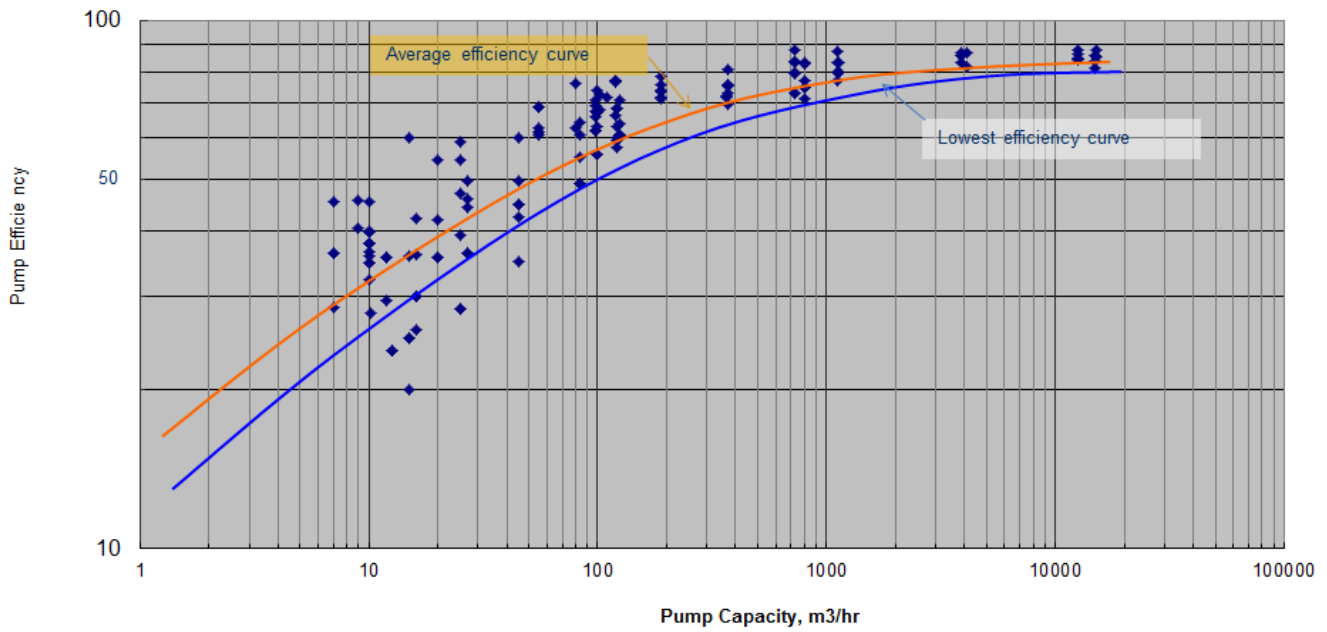
Image of impeller

| | Single suction | | |
|--|----------------|------------|---|
| Flow coef., Cq | 0.15 | 0.15 | Input, 0.05-0.1 for small pump, 0.1 - 0.2 for larger pump |
| No of blade, Z | 6 | 6 | Input |
| Blade thickness (mm), t | 5 | 5 | Input |
| Capacity, Qms (m ³ /s) | 0.45 | 0.45 | Qms = Capacity in m ³ /s |
| Head coef., Y | 0.55 | 0.55 | |
| Tangential vel. U (m/s) | 24.89 | 24.89 | |
| Head each impelle., H (m) | 34.74 | 34.74 | |
| Cu ² (m/s) as per selected Y | 13.69 | 13.69 | Cu = Y * U |
| Radial vel as per selected Cq, Vr (m/s) | 3.73 | 3.73 | Vr = Cq * U |
| (U-Cu) | 11.20 | 11.20 | |
| Vr / (U-Cu) | 0.333 | 0.333 | |
| Impeller tip dia, D (mm) | 485.1 | 485.1 | |
| Exit angle (Beta) after slip (deg) | 18 | 18 | |
| First trial Slip factor, S1 | 0.770 | 0.770 | |
| First trial Cu' 1(m/s) | 17.79 | 17.79 | |
| First trial Vr / (U-Cu)1 | 0.525 | 0.525 | |
| First trial Exit angle (Beta1) before slip (deg) | 27.7 | 27.7 | |
| Slip factor, S | 0.728 | 0.728 | Wiesner eq. |
| Cu (m/s) | 18.81 | 18.81 | |
| Vr / (U-Cu) | 0.61 | 0.61 | Backward up to radial exit |
| Exit angle (Beta) before slip (deg) | 32 | 32 | |
| Radial blade thickness, Lb (mm) | 57.3 | 57.3 | Lb = Z * T / sin (beta) |
| Exit area, A2 = Q/Vr (m ²) | 0.12051528 | 0.12051528 | |
| Exit width, b (mm) | 82.2 | 82.2 | |

Cq will affect to exit angle and width of impeller tip exit design. Cq = Vr / U

EFFICIENCY ESTIMATION BASED ON PUMP CAPACITY

Centr pump efficiency, single stage



Average efficiency formula (approached, in %):

Below 1000 m3/h, Eff. (%) = $9.9943 \times \ln(Q) + 10.535$

1000 m3/h and above, Eff. (%) = $2.6178 \times \ln(Q) + 60.972$

(1)

EFFICIENCY ESTIMATION BASED ON PUMP SPECIFIC SPEED

Specific speed, $N_s = N \cdot Q^{0.5} / H^{0.75}$

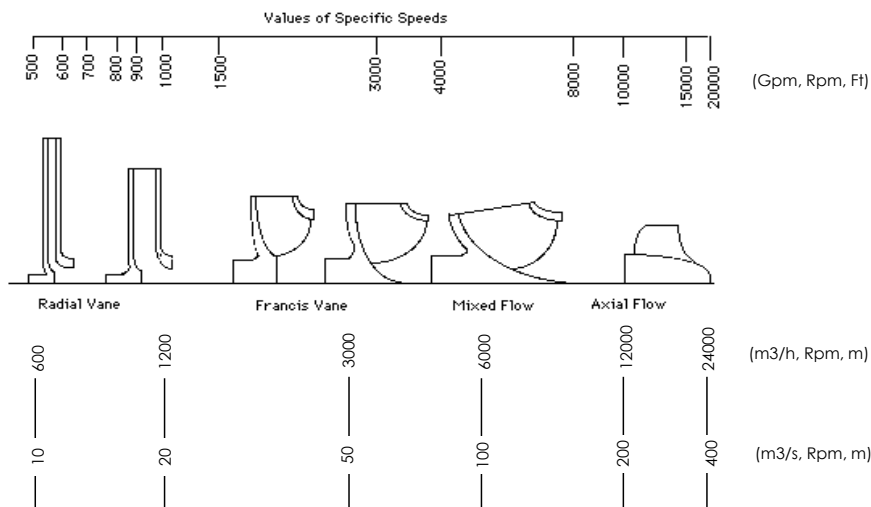
N Rotation speed in RPM

Q Capacity in GPM or in m3/h

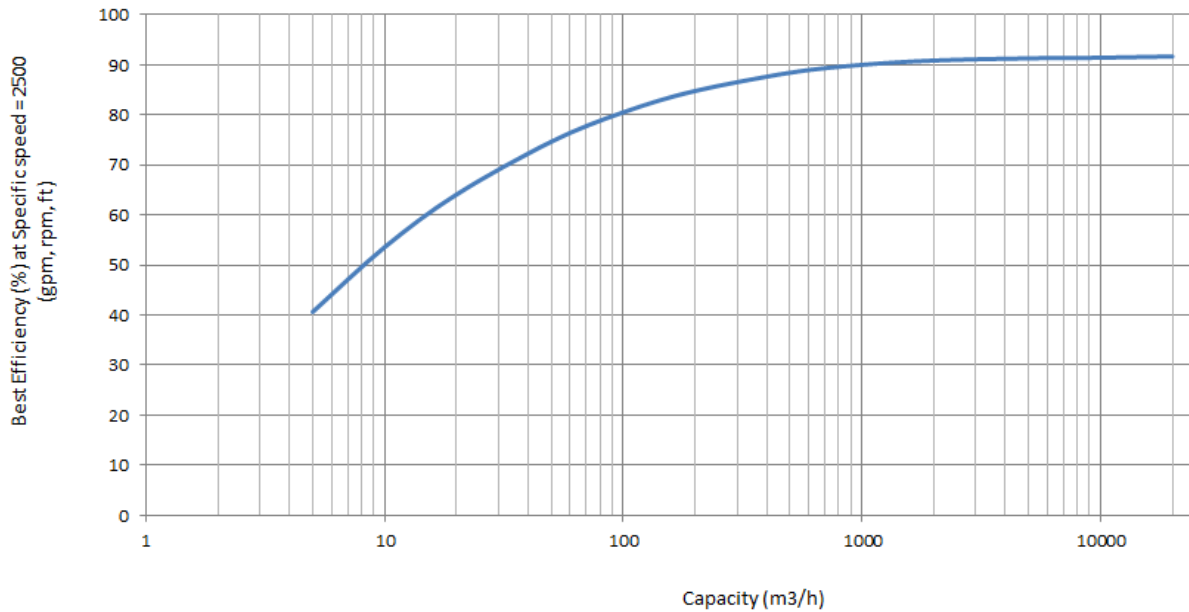
H Head in Ft or in meter

Ns Specific speed can be in (GPM, RPM, FT) or in (m3/h, RPM, m)

(2)



BEST EFFICIENCY

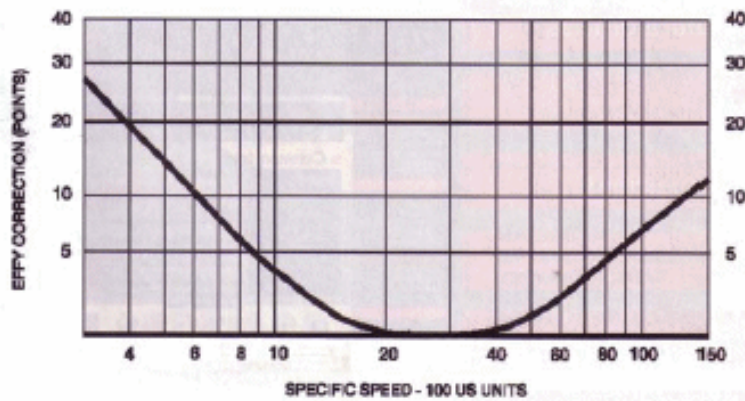


Best efficiency assume at $N_s = 2500$ (GPM, RPM, FT) can be expressed in formula:

$$E_{fb} (\%) = 0.1279(\ln(Q))^3 - 3.4119(\ln(Q))^2 + 30.521(\ln(Q)) - 5.1945, \text{ if } Q \text{ in } m^3/h \tag{3}$$

$$E_{fb} (\%) = 0.1279(\ln(0.2771Q))^3 - 3.4119(\ln(0.2771Q))^2 + 30.521(\ln(0.2771Q)) - 5.1945, \text{ if } Q \text{ in GPM}$$

Efficiency shall be corrected if N_s not at 2500 (gpm, rpm, ft) by following curve.



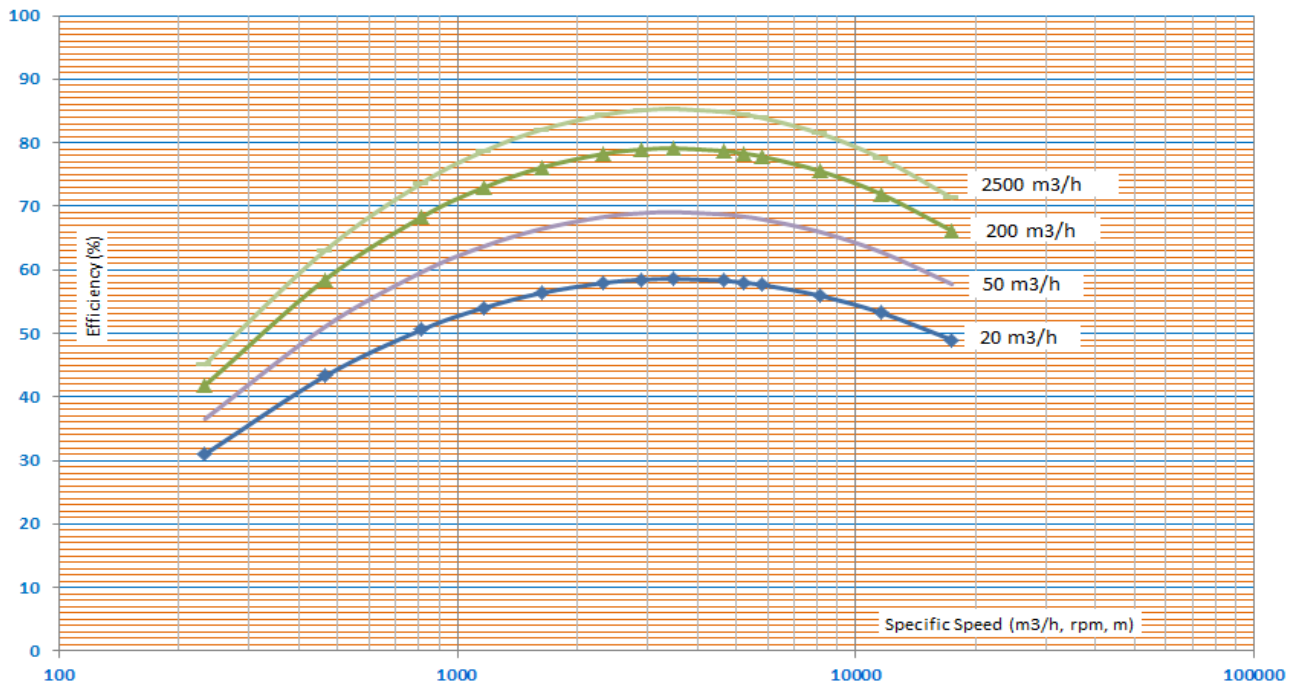
This curve can be expressed in formula:

$$E_{fcor} (\text{point}) = 6.3328 * (\ln(0.861N_s))^2 - 101.5029 * \ln(0.861N_s) + 407.5364, \text{ if } N_s \text{ in } (m^3/h, RPM, m) \tag{4}$$

$$E_{fcor} (\text{point}) = 6.3328 * (\ln(N_s))^2 - 101.5029 * \ln(N_s) + 407.5364, \text{ if } N_s \text{ in } (GPM, RPM, Ft)$$

Corrected efficiency become:

$$E_f = E_{fb} (1 - 0.01 E_{fcor}) \tag{5}$$



FOLLOWING EXAMPLE IS APPLICABLE FOR SINGLE IMPELLER (SINGLE STAGE) PUMP.

INPUT :

| | | |
|--------------------|------|---|
| Capacity (Q, m3/h) | 150 | This head shall be for single impeller only |
| Head (H, m) | 32 | |
| Speed (N, rpm) | 2980 | |

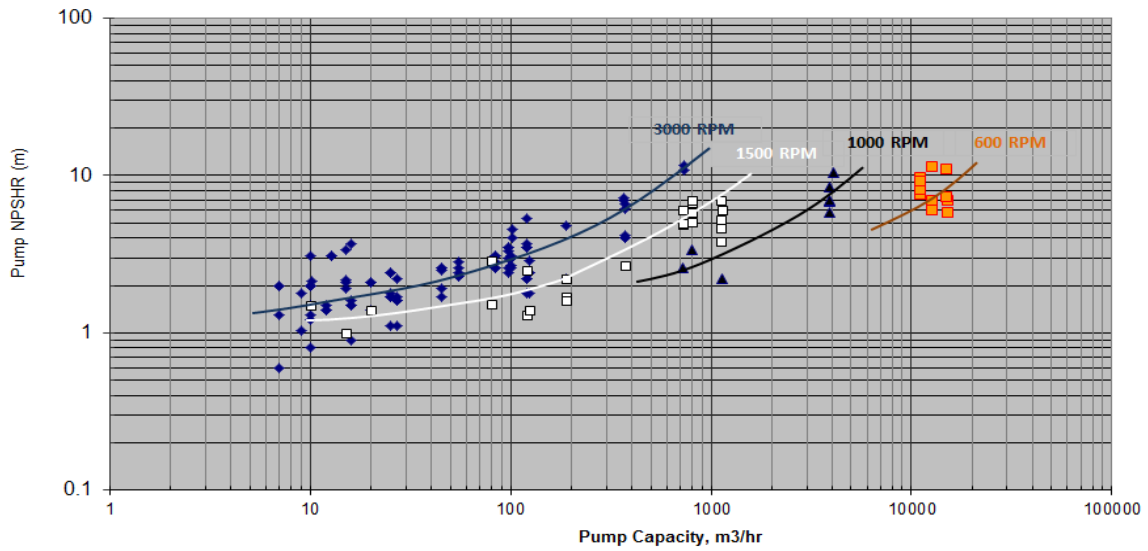
OUTPUT :

| | |
|-------------------------|---------|
| Spec. speed (Ns) | 2712.69 |
| Best eff (Efb, %) | 78.16 |
| Eff Corr (Efcor, point) | 1.23 |
| Efficiency (Ef, %) | 77.20 |
| Avrg eff as per eq (1) | 60.61 |

$Specific\ speed,\ Ns = N \cdot Q^{0.5} / H^{0.75}$
 $Efb\ (\%) = 0.1279(\ln(Q))^3 - 3.4119(\ln(Q))^2 + 30.521(\ln(Q)) - 5.1945$, if Q in m3/h
 $Efcor\ (point) = 6.3328 * (\ln(0.861Ns))^2 - 101.5029 * \ln(0.861Ns) + 407.5364$, if Ns in (m3/h, RPM, m)
 $Ef = Efb (1 - 0.01Efcor)$

FOR MULTI STAGE SEE OTHER SHEETS ("Efficiency & NPSH", "More detail horizontal" and "More detail vertical")

Centr pump NPSHR



Approached formula for NPSHR (Average, in meter) = $Q \times (5.8082 \times 10^{-10} \times N^2 + 2.7012 \times 10^{-6} \times N - 1.4161 \times 10^{-3}) + \{0.501 \times \ln(N) - 2.237\}$
 Where Q in m3/h and N in RPM

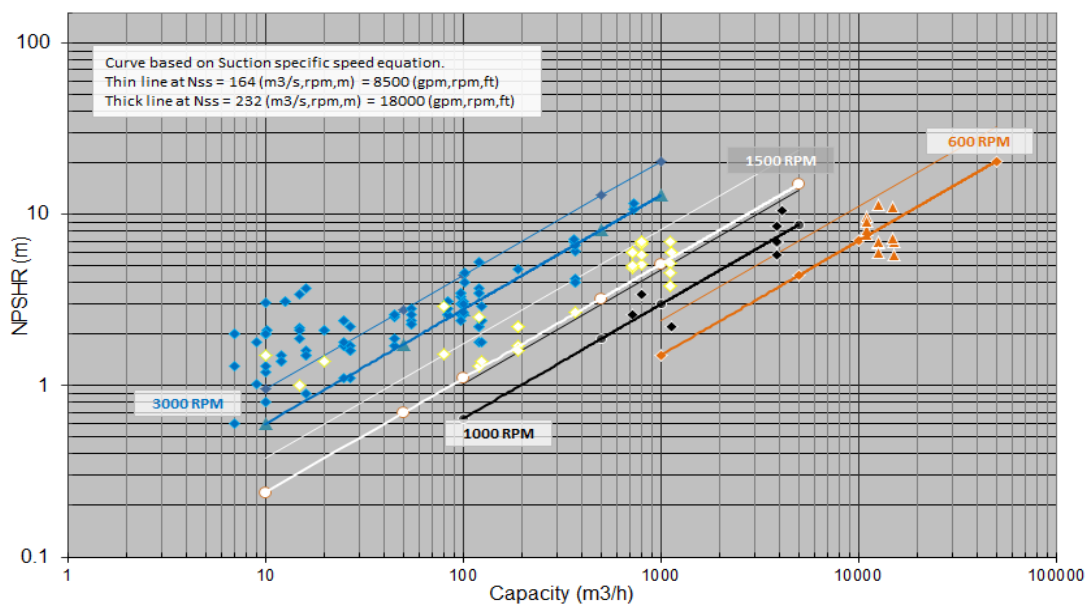
NPSHR estimation using Suction Specific Speed, Nss

Suction Specific Speed, Nss ($= N \cdot Q^{0.5} / \text{NPSHR}^{0.75}$)

| Suction Specific Speed, Nss | | | Category or level | How to reduce or select lower NPSHR |
|-----------------------------|--------------|--------------|-------------------|--|
| m3/s,rpm,m | (gpm,rpm,ft) | (m3/h,rpm,m) | | |
| <164 | <8500 | <9874 | very high NPSHR | Using formula $\text{NPSHR} = (N \cdot Q^{0.5} / \text{Nss})^{1.333}$: 1. Select lower speed (lower N) when NPSHR is not available 2. Lower speed cause bigger diameter. When bigger diameter is not preferred, select double suction 3. When double suction is not preferred, use inducer or can be vertical type when suction level is limited or combined |
| 165 | 8500 | 9874 | high NPSHR | |
| 232 | 12000 | 13940 | medium NPSHR | |
| 349 | 18000 | 20910 | low NPSHR | |
| 465 | 24000 | 27880 | very low NPSHR | |

Below chart based on formula $\text{NPSHR} = (N \cdot Q^{0.5} / \text{Nss})^{1.333}$

Centrifugal Pump NPSHR



MANUALY ESTIMATION FOR CENTRIFUGAL PUMP NPSHR

INPUT:

Capacity (Q, m³/h)
Speed (N, rpm)

| |
|------|
| 150 |
| 2980 |

| |
|------|
| 75 |
| 2980 |

OUTPUT:

| | Single suction | | | Use average formula | Double suction | | | Use average formula | Single suction with inducer | | |
|-------------|------------------------------------|--------------|----------------|---------------------------|------------------------------------|------------------|-------|---------------------------|------------------------------------|-------|--|
| | at Nss (m ³ /h, rpm, m) | | | | at Nss (m ³ /h, rpm, m) | | | | at Nss (m ³ /h, rpm, m) | | |
| | | 10000 | 15000 | | 10000 | 15000 | 20000 | | | 27000 | |
| | at Nss (m ³ /s, rpm, m) | | | | at Nss (m ³ /s, rpm, m) | | | | at Nss (m ³ /s, rpm, m) | | |
| | | 167 | 250 | | 167 | 250 | 333 | | | 450 | |
| | Highest NPSH | High NPSH | Middle NPSH | Middle NPSH | Low NPSH | Very Low NPSH | | Very Low NPSH | | | |
| NPSHR (m) = | | 5.62 | 3.27 | 3.54 | 3.54 | 2.06 | 1.40 | 2.66 | Can be: | 1.49 | |

EFFICIENCY, NPSHR, SPEED, AND TIP DIAMETER ESTIMATION

Input data to be filled in red and brown cells only

This calculation method is applied in "More detail horizontal" and "More detail vertical" worksheets

| Description | Unit | Quantity | Note, remark |
|---|--------------------------|--------------------------------|--|
| Capacity, Q | m ³ /h | 300 | |
| Head, H | m | 64 | |
| NPSHA | m | 9.2 | |
| Margin for NPSH | m | 1 | |
| NPSHR shall be lower than | m | 8.2 | |
| Trial Speed, N (example: when driven by motor 2980, 1480, 980, 730, 580 for 50 Hz motor, or 3580, 1780, 1180, 890, 700 for 60 Hz motor: | RPM | 3500 | Selected and trialed speed shall give one NPSH in table below smaller than required NPSHR. For turbine driven or with increasing gear, speed may higher than motor driven direct couple. |
| Check whether selected speed still give NPSHR below (NPSHA-margin), (Y/N) | | Y | |
| Possibility suction type (1 st impeller) | | Single Suction without inducer | |
| Select suction type using code number <i>Fill with: A. Single suction without inducer, B. Double Suction, C. Single suction with inducer</i> | | A | |
| Possibility NPSHR, lower | m | 6.44 | |
| higher | m | 11.05 | |
| highest | m | | |
| Select and trial max. tip speed, U | m/s | 35 | |
| Design head coefficient, Yd | | 0.55 | |
| Check whether selected U and Yd is OK | | OK | |
| Max. head per impeller, Hmax=Y.U ² /g where Y = Cu/U | m | 68.7 | |
| Number of impeller | | 1 | |
| Average head eah impeller, Havg | m | 64.00 | |
| Average tip diameter | mm | 191.0 | |
| Head coefficient as per required condition, Y | | 0.513 | |
| Capacity at each impeller eye of 1 st impeller (half of capacity for double suction) | m ³ /h | 300 | |
| Specific speed of first impeller | m ³ /h, rpm,m | 2679 | |
| Calculated 1 st impeller efficiency | % | 80.58 | |
| Specific speed of next impellers | m ³ /h, rpm,m | 0.00 | |
| Calculated next impellers efficiency | % | 0.00 | |
| Average efficiency | % | 80.58 | |

| Single suction | | Double suction | | Single suction with inducer | |
|------------------------------------|-----------|------------------------------------|-------------|------------------------------------|---------------|
| at Nss (m ³ /h, rpm, m) | | at Nss (m ³ /h, rpm, m) | | at Nss (m ³ /h, rpm, m) | |
| | 10000 | 15000 | 15000 | 20000 | 27000 |
| at Nss (m ³ /s, rpm, m) | | at Nss (m ³ /s, rpm, m) | | at Nss (m ³ /s, rpm, m) | |
| | 167 | 250 | 250 | 333 | 450 |
| Highest NPSH | High NPSH | Middle NPSH | Middle NPSH | Low NPSH | Very Low NPSH |
| | 11.05 | 6.44 | | 4.06 | 2.76 |
| | | | | Can be: | 2.94 |

Select untill Hmax near but higher than Havg
Head coefficient, most data: 0.4 up to 0.6, except low speed to provide low NPSHR may near 1 (radial exit)

| Tip speed selection | m/s | |
|--|-----|------------------------------|
| Pumps fitted with elastomer impellers | 26 | |
| Higher slurry concentrations and larger solids | 30 | |
| Medium slurries up to 25% solids and 200 micron solids | 35 | |
| Dirty water | 40 | Sea water, river water, etc. |
| Clean liquid & hot with suitable mat'l | 55 | BFW, clean process |
| Clean liquid cold & warm with suitable mat'l | 85 | Demin water, clean process |

Head coefficient : $Y = C_u / U$
Flow coefficient : $C_q = V_r / U$

Velocity triangle

THIS CALCULATION METHOD IS APPLIED IN "More detail horizontal" and "More detail vertical" worksheets